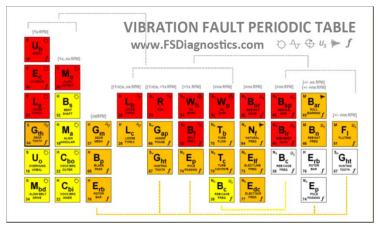
# **VIBRATION ANALYSIS CASE HISTORY**





Dan Ambre is the founder of Full Spectrum Diagnostics, (June of 2000) providing the finest in Machinery Condition Monitoring, Specialized Analyses, and Vibration Training for Industrial, Manufacturing, Aerospace, and Automotive applications.

Dan is a graduate of *The University of Iowa* with a Bachelor's degree in Mechanical Engineering, and has completed additional graduate level course work in Engineering Dynamics from *The University of Illinois at Chicago*, and *Florida Atlantic University*.

Dan has over 30 years experience in the aviation and aerospace industries with a background in Vibration & Acoustic testing, Dynamic analysis of high speed rotor systems, Experimental Modal, and Finite Element Analysis. His consulting vibration experience base comes from positions at Sundstrand Aviation Corporation, Pratt & Whitney (United Technologies Corporation), and Technical Associates of Charlotte.

Dan is a Certified Level III / CAT IV. Vibration Training Instructor and a registered Professional Engineer in Minnesota.



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

#### **PROBLEM STATEMENT:**

This machine was experiencing Elevated Vibration response at 1x RPM. The dominant response was in the Vertical Direction on the outboard bearing support.

#### **PRELIMINARY ASSESSMENT:**

- 1. Unbalance
- 2. Coupling Alignment Error (Vertical Offset)
- 3. Bearing Assembly Alignment Error
- 4. Natural Frequency excited by Pump 1x RPM response.
- 5. Amplified Piping Response

#### **FINAL CONCLUSIONS:**

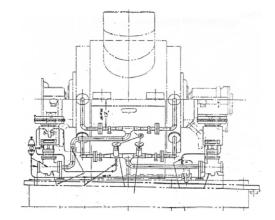
 Natural Frequency excited by Pump Residual 1x RPM response.

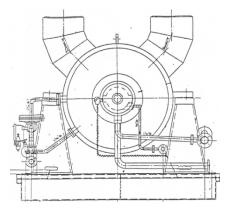
### **RECOMMENDATIONS / OPTIONS:**

- Change Rotor Speed to Avoid Resonance
- Add Mass: Reduce Natural Frequency
- Remove Mass: Increase Natural Frequency
- Add Stiffness: Increase Natural Frequency
- Remove Stiffness: Reduce Natural Frequency
- Add Damping: Reduce Resonance Amplification









Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

#### **OVERALL RESPONSE:**

Overall response showed "step" changes in response in the Vertical Vibration levels at the pump outboard bearing housing. The step changes were associated with maintenance outage unit restarts.

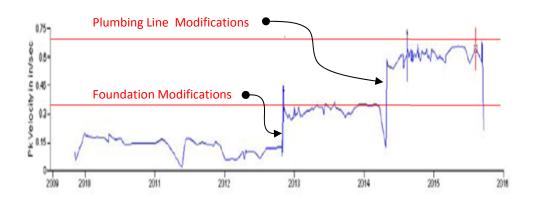
Notations indicate significant structural modifications to this unit. There appears to be a shift in amplitude that is associated with each recent change.

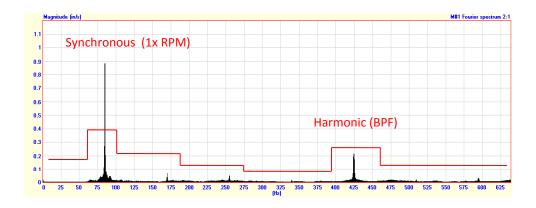
#### **SPECTRUM RESPONSE:**

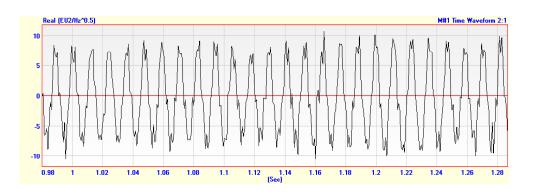
The Spectrum response shown indicates a dominant 1x rotor speed (5,100 RPM / 85 Hz) vertical vibration level at the pump outboard bearing location. Also shown is the Blade Passing Frequency (BPF) at 5x RPM.

#### **TIME WAVEFORM RESPONSE:**

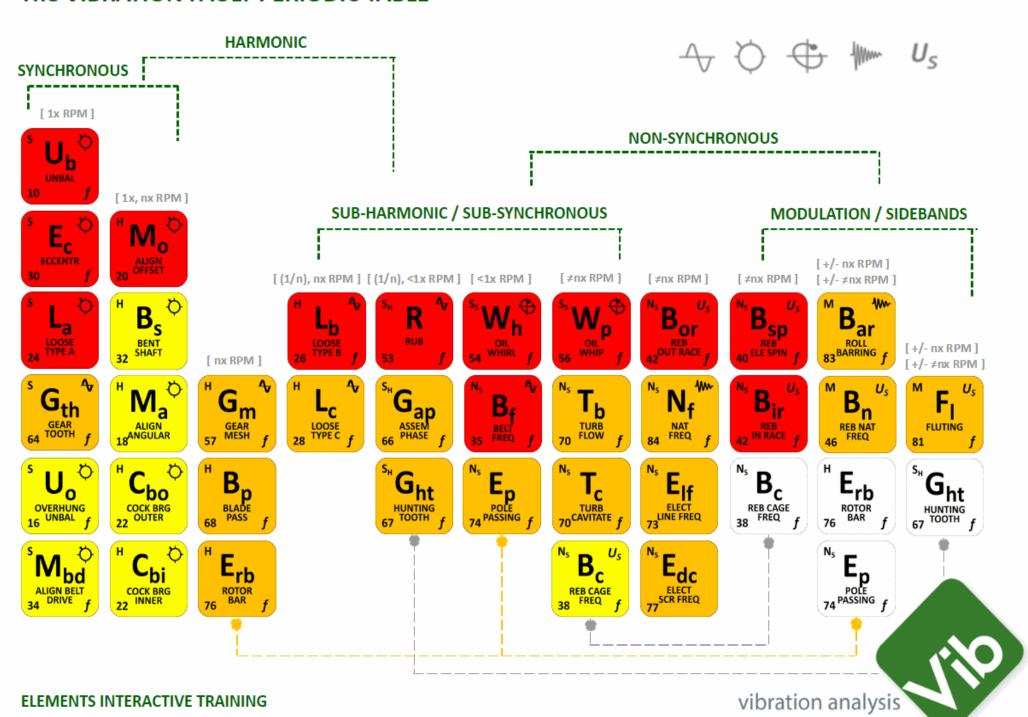
The Time Waveform shows a dominant synchronous response with a slight higher frequency distortion due to the 5x RPM blade passing frequency.

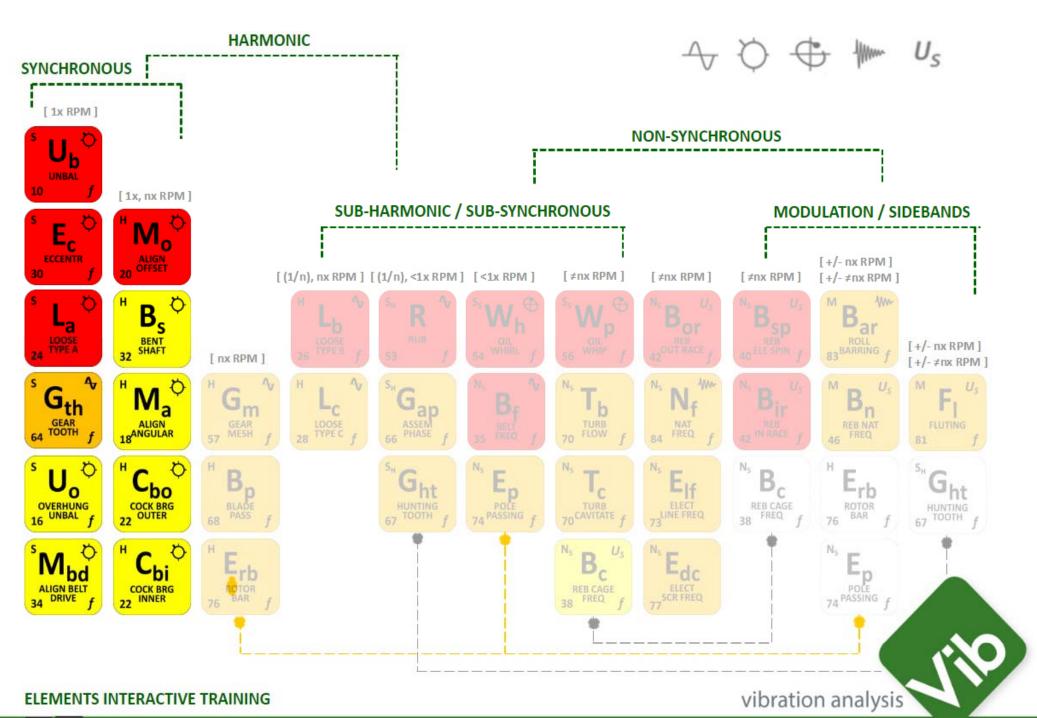


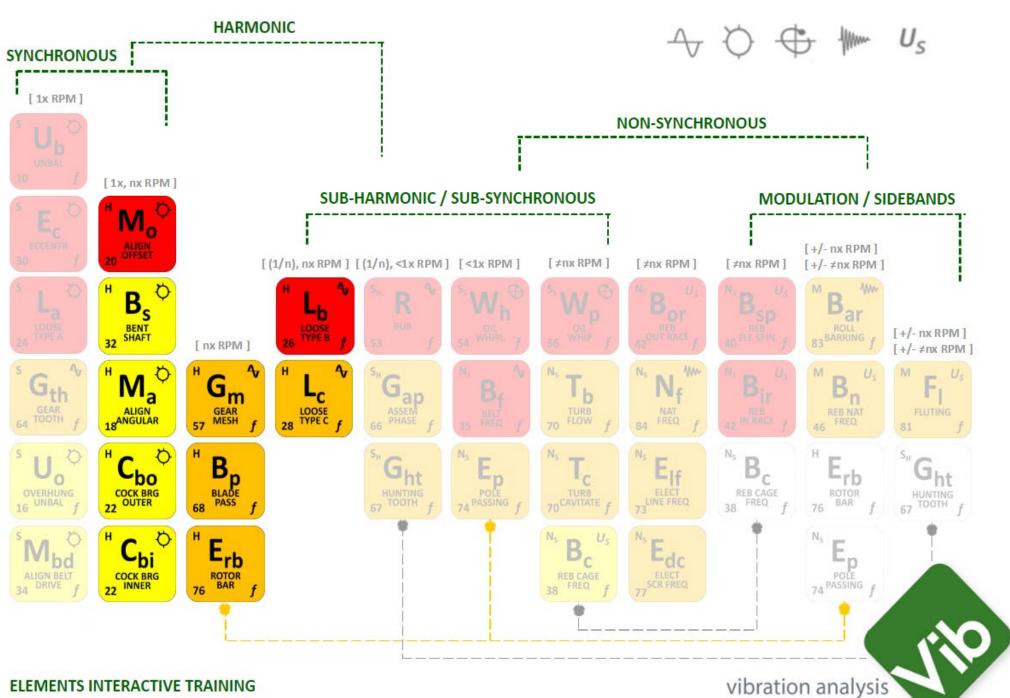


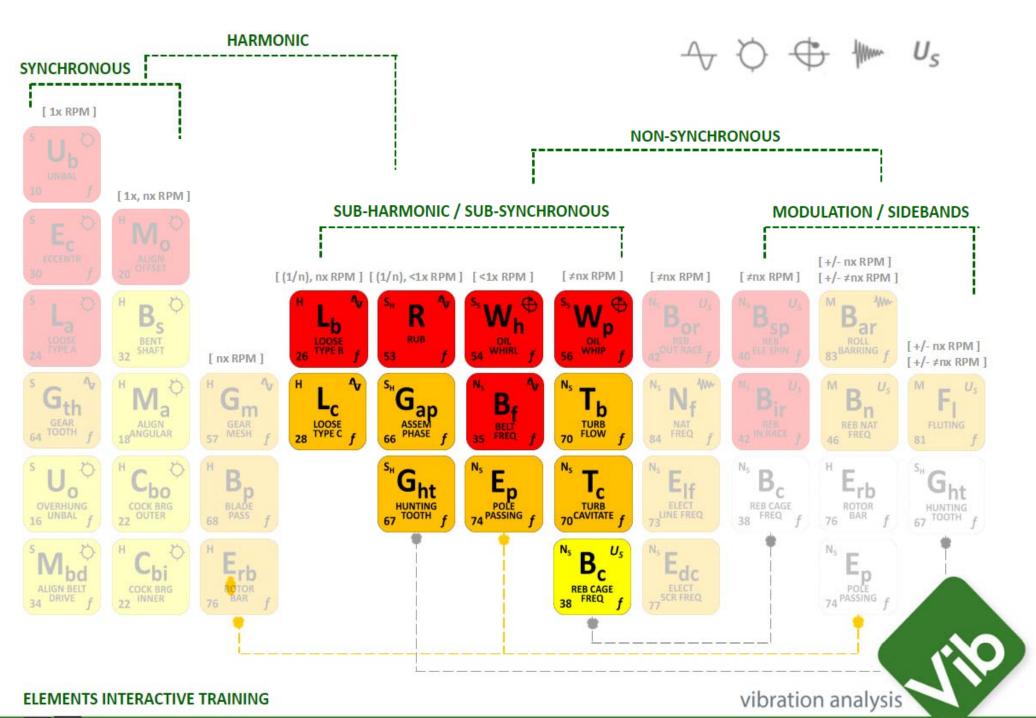


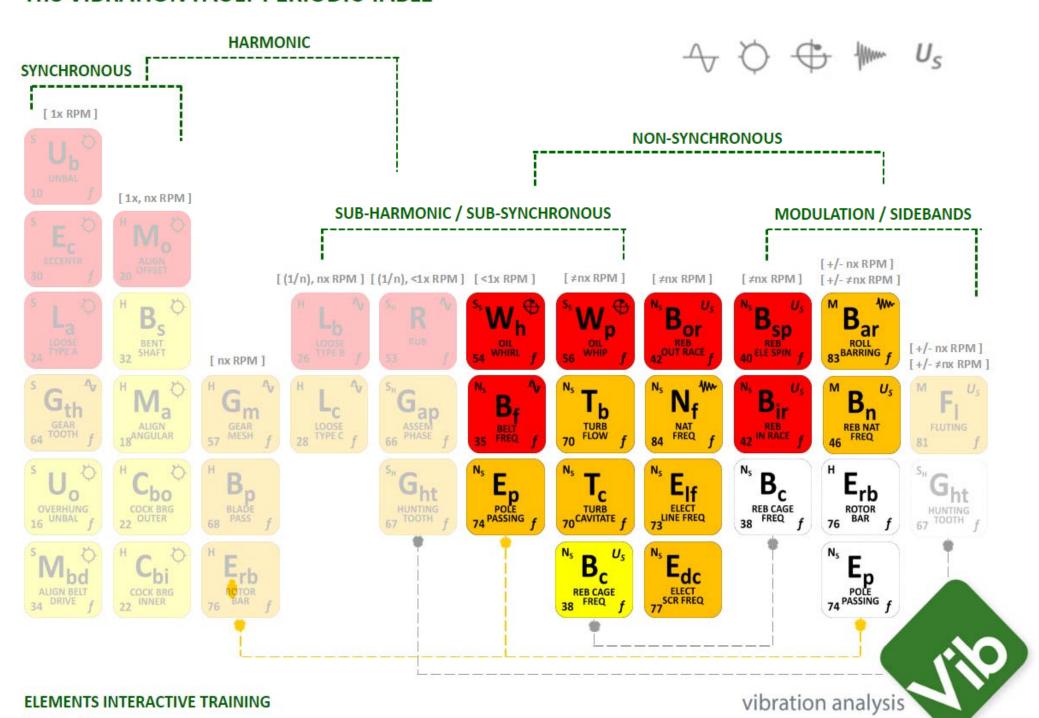


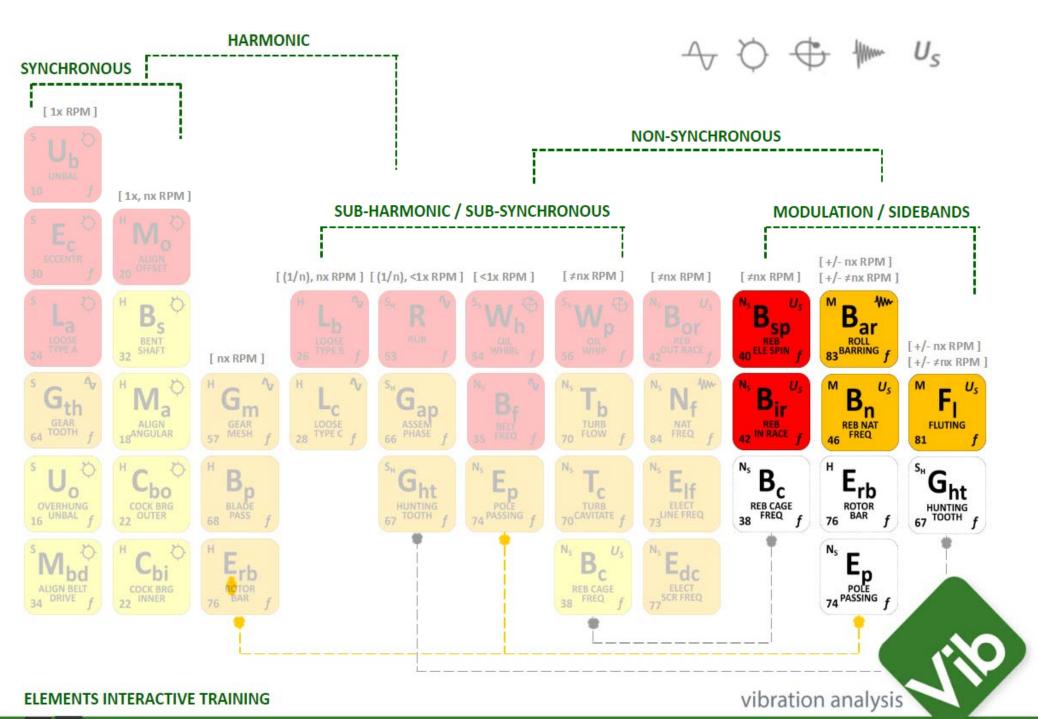


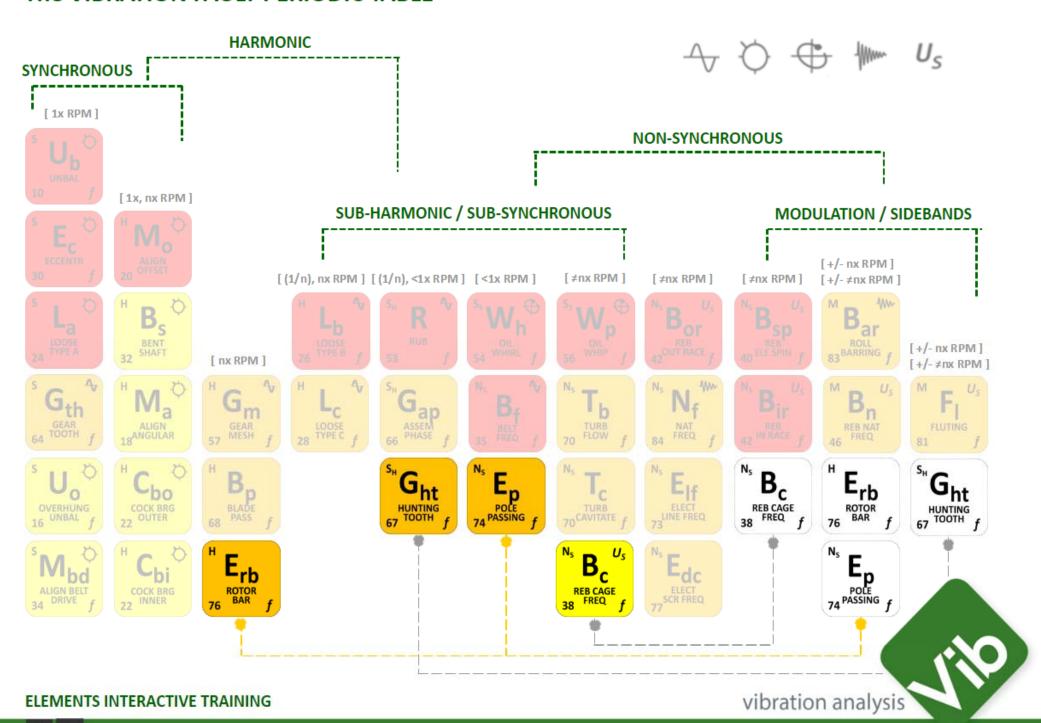


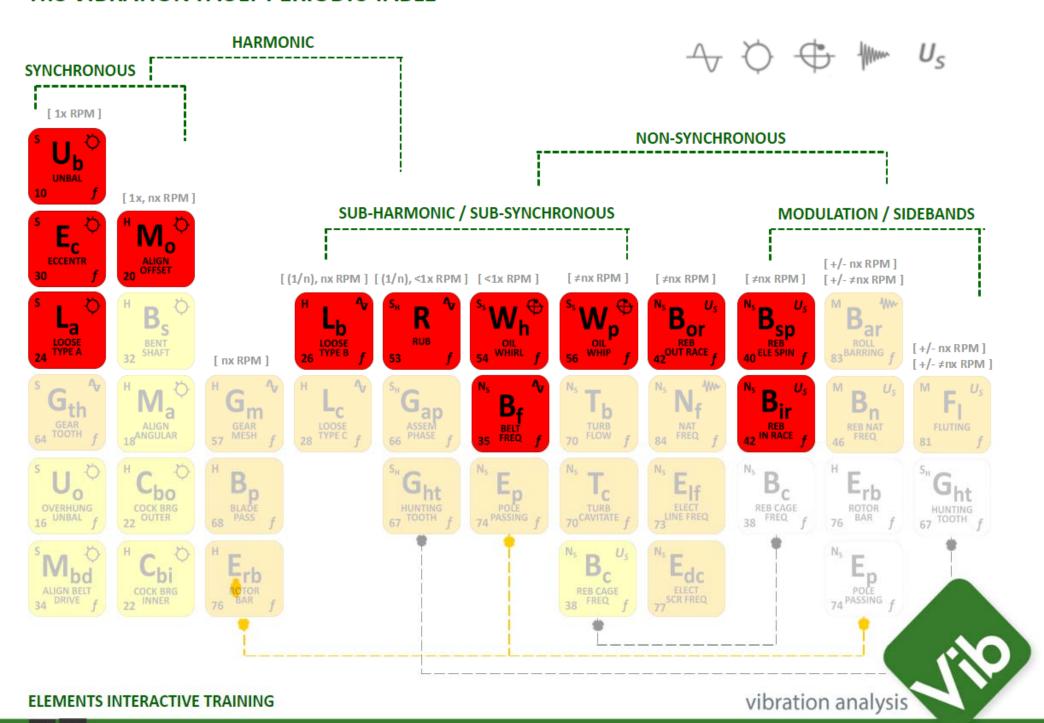


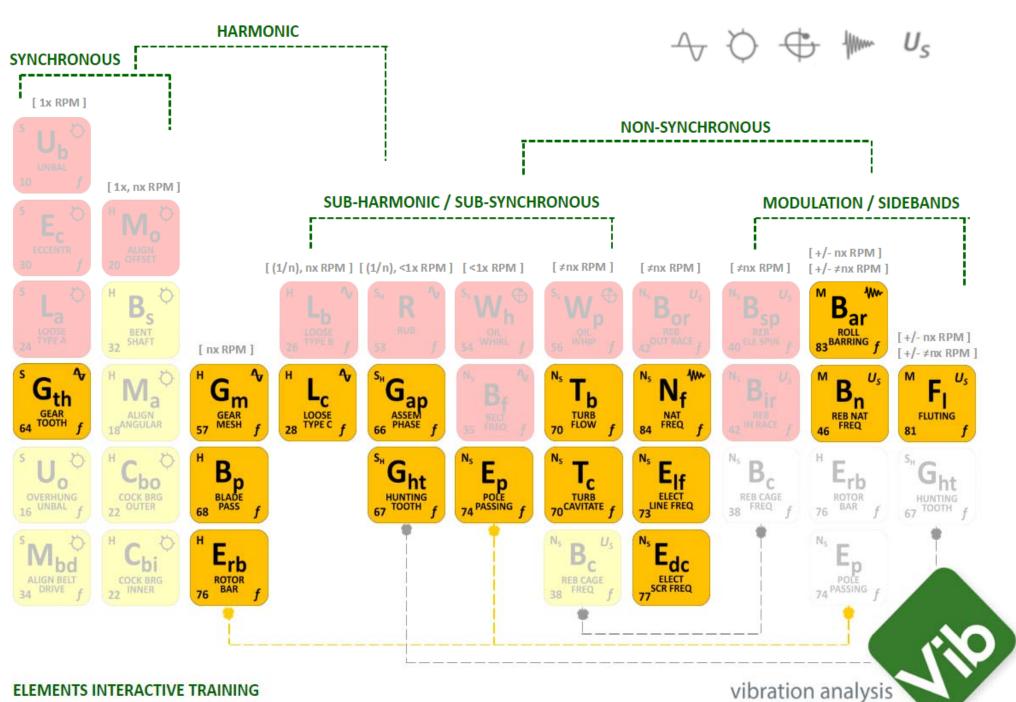


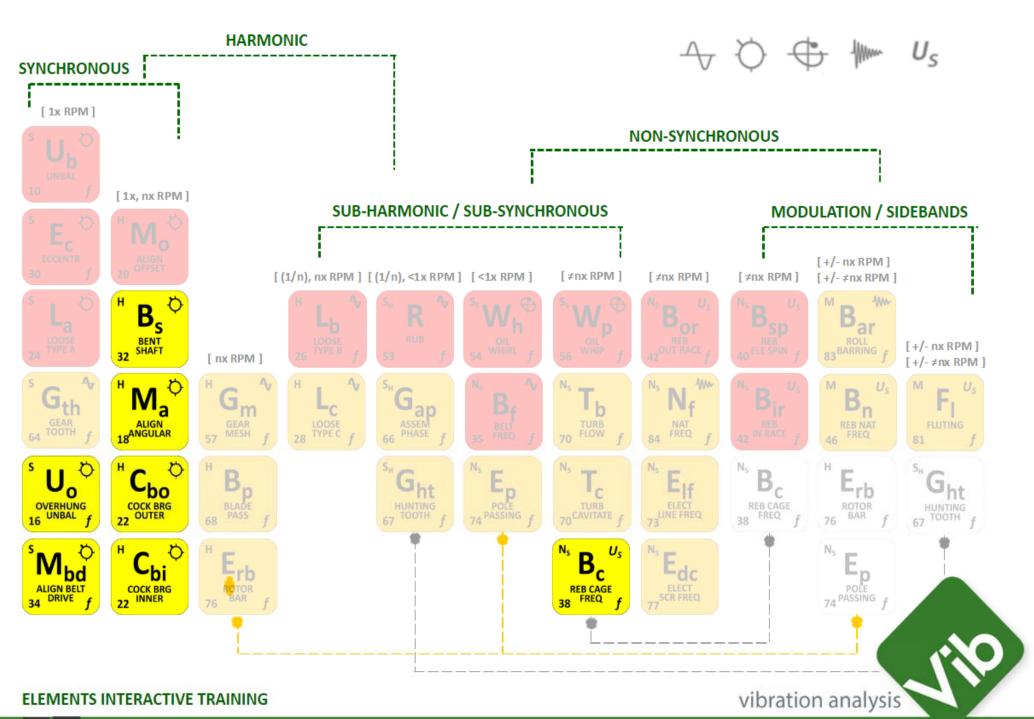












Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

### THE VIBRATION FAULT PERIODIC TABLE

# **ANALYSIS & TESTING**

## **SYNCHRONOUS VIBRATION** [1x RPM]

**Potential Causes:** 

**U<sub>h</sub>** Unbalance

**E**<sub>c</sub> Eccentricity

L<sub>a</sub> Mechanical Looseness Type A

**G**<sub>th</sub> Gear Tooth Fault

U<sub>o</sub> Overhung Rotor Unbalance

**M**<sub>bd</sub> Belt Drive Misalignment

**M**<sub>o</sub> Offset Coupling Misalignment

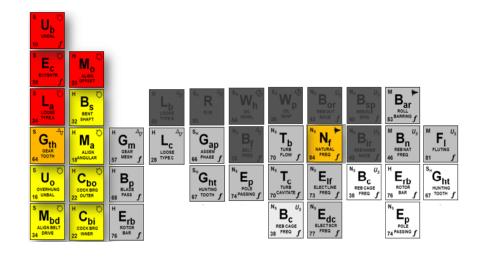
**B**<sub>s</sub> Bent / Bowed Shaft

**M**<sub>a</sub> Angular Coupling Misalignment

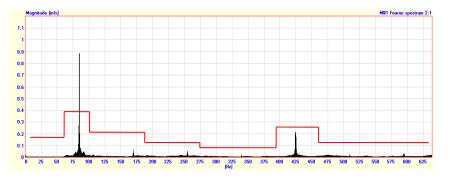
**C**<sub>bo</sub> Cocked Bearing Outer Race

**C**<sub>bi</sub> Cocked Bearing Inner Race

**N<sub>f</sub>** Natural Frequency / Resonant Amplification



### **SOURCES of SYNCHRONOUS VIBRATION**



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## THE VIBRATION FAULT PERIODIC TABLE

# **ANALYSIS & TESTING**

# **SYNCHRONOUS VIBRATION** [1x RPM]

**Potential Causes:** 

**U<sub>h</sub>** Unbalance

**E**<sub>c</sub> Eccentricity

L<sub>a</sub> Mechanical Looseness Type A

**G**<sub>th</sub> Gear Tooth Fault

U<sub>o</sub>-Overhung Rotor Unbalance

M<sub>bd</sub>—Belt Drive Misalignment

**M**<sub>o</sub> Offset Coupling Misalignment

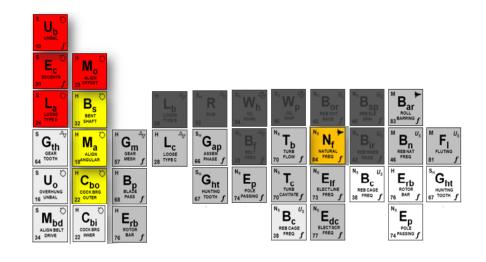
**B**<sub>s</sub> Bent / Bowed Shaft

M<sub>a</sub> Angular Coupling Misalignment

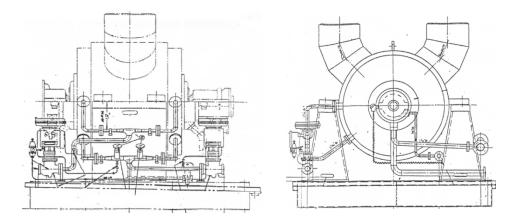
C<sub>bo</sub> Cocked Bearing Outer Race

**G**<sub>bi</sub>— **Cocked Bearing Inner Race** 

**N**<sub>f</sub> Natural Frequency / Resonant Amplification



### BY DESIGN SOME FAULTS ARE RULED-OUT



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

### THE VIBRATION FAULT PERIODIC TABLE

# **ANALYSIS & TESTING**

## **SYNCHRONOUS VIBRATION** [1x RPM]

**Potential Causes:** 

Unbalance

E<sub>c</sub>-Eccentricity

# L<sub>a</sub> Mechanical Looseness Type A

**G**<sub>th</sub> Gear Tooth Fault

U<sub>o</sub> Overhung Rotor Unbalance

M<sub>bd</sub> Belt Drive Misalignment

# **M**<sub>o</sub> Offset Coupling Misalignment

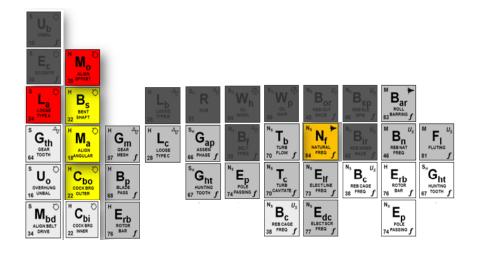
**B**<sub>s</sub> Bent / Bowed Shaft

**M**<sub>a</sub> Angular Coupling Misalignment

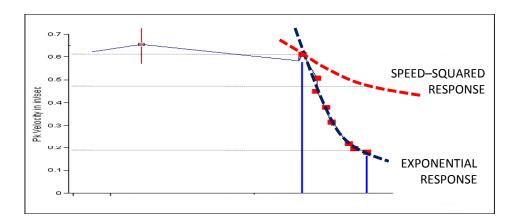
**C**<sub>bo</sub> Cocked Bearing Outer Race

C<sub>hi</sub> Cocked Bearing Inner Race

**N<sub>f</sub>** Natural Frequency / Resonant Amplification



#### SOME FAULTS ARE RULED-OUT BY TRANSIENT BEHAVIOR



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

### THE VIBRATION FAULT PERIODIC TABLE

# **ANALYSIS & TESTING**

## **SYNCHRONOUS VIBRATION** [1x RPM]

**Potential Causes:** 

U<sub>h</sub> Unbalance

E<sub>c</sub> Eccentricity

# L<sub>a</sub> Mechanical Looseness Type A

**G**<sub>th</sub> Gear Tooth Fault

**U**<sub>o</sub> Overhung Rotor Unbalance

M<sub>bd</sub> Belt Drive Misalignment

# **M**<sub>o</sub> Offset Coupling Misalignment

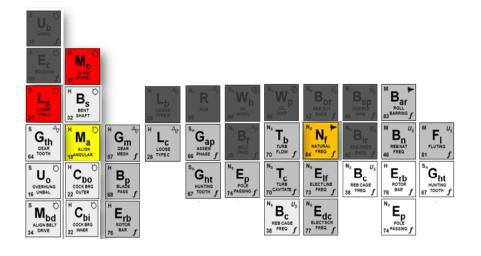
B<sub>s</sub>—Bent / Bowed Shaft

M<sub>a</sub> Angular Coupling Misalignment

**C**<sub>bo</sub>- Cocked Bearing Outer Race

C<sub>bi</sub> Cocked Bearing Inner Race

**N**<sub>f</sub> Natural Frequency / Resonant Amplification



#### SOME FAULTS ARE RULED-OUT BY INSPECTION







Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# THE VIBRATION FAULT PERIODIC TABLE

# **ANALYSIS & TESTING**

## **SYNCHRONOUS VIBRATION** [1x RPM]

**Potential Causes:** 

U<sub>h</sub> Unbalance

E<sub>c</sub> Eccentricity

G<sub>th</sub> Gear Tooth Fault

U<sub>o</sub> Overhung Rotor Unbalance

M<sub>bd</sub> Belt Drive Misalignment

M. Offset Coupling Misalignment

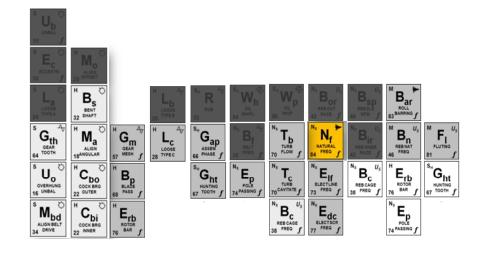
B<sub>s</sub> Bent / Bowed Shaft

M<sub>a</sub>- Angular Coupling Misalignment

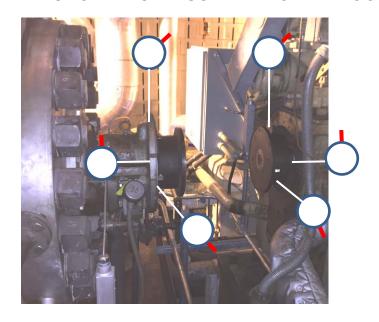
C<sub>bo</sub> Cocked Bearing Outer Race

C<sub>bi</sub> Cocked Bearing Inner Race

N<sub>f</sub> Natural Frequency / Resonant Amplification



#### SOME FAULTS ARE RULED-OUT BY PHASE ANALYSIS





Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

### THE VIBRATION FAULT PERIODIC TABLE

# **ANALYSIS & TESTING**

## **SYNCHRONOUS VIBRATION** [1x RPM]

**Potential Causes:** 

U<sub>h</sub> Unbalance

**E**<sub>c</sub> Eccentricity

L<sub>a</sub> Mechanical Looseness Type A

**G**<sub>th</sub> **Gear Tooth Fault** 

U<sub>o</sub> Overhung Rotor Unbalance

M<sub>bd</sub> Belt Drive Misalignment

M. Offset Coupling Misalignment

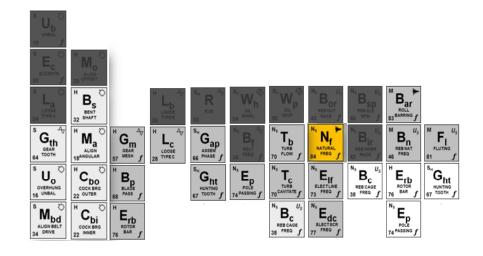
**B**<sub>s</sub> Bent / Bowed Shaft

M<sub>a</sub> Angular Coupling Misalignment

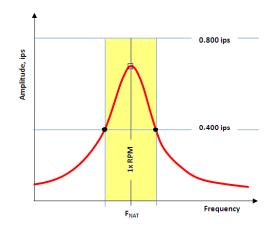
C<sub>bo</sub> Cocked Bearing Outer Race

C<sub>hi</sub> Cocked Bearing Inner Race

**N**<sub>f</sub> Natural Frequency / Resonant Amplification



### **MOST LIKELY FAULT is IDENTIFIED**



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# FEED WATER PUMP HISTORY

**SYNCHRONOUS VIBRATION** [1x RPM]

**Potential Causes:** 

Natural Frequency / Resonant Amplification

**Current Motion Indicates Vertical Pitching Mode** 

2005

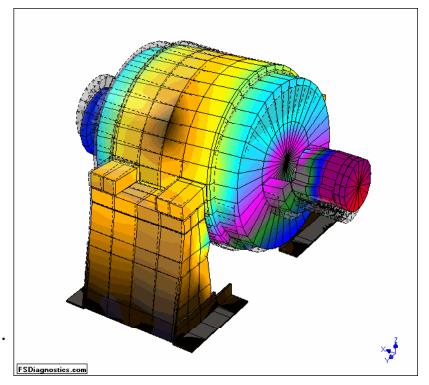
EMA defined HORIZONTAL Torsional Mode @ 5,112 RPM [1x]. This Resonance is Resolved by Structural Modification

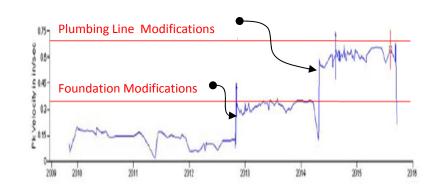
2005

EMA defined a VERTICAL PITCHING MODE @ 4,200 CPM [>20% Resonance Margin] Recommended Pedestal Structural Modification is NOT expected to alter this mode.

### 2015

The 2005 Vertical Pitching Mode at 4,200 CPM is suspected to be stiffened into the Operating Speed Range due to Large Piping Line (2014) and/or Foundation Modifications (2012) associated with the shifts in Vertical Overall Vibration response.







Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# **ANALYSIS & TESTING PROPOSAL**

**REVIEW** 2005 Experimental Modal Analysis

Data / Analysis / Methodologies / Assumptions / Conclusions

**PERFORM** Operating Modal Analysis (OMA) on FW-P-32B Determine Current Natural Frequencies, Mode Shapes

**CONTRUCT** Finite Element Analysis (FEA) Model of Pump
Use 2005 EMA and 2015 OMA Analyses for Correlation of Modes and Mode Shapes

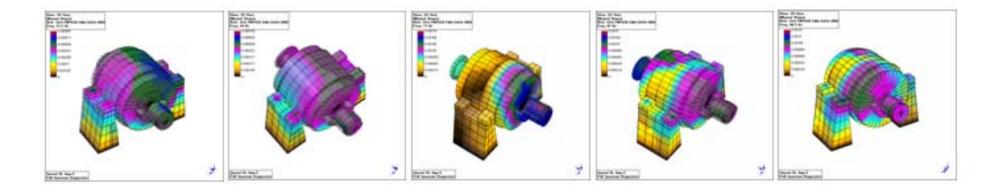
Use FEA Model to **DEFINE** Structural Modifications to Eliminate or Reduce 1x RPM Vibration Response Stiffness / Mass / Damping / Phase Cancellation

VERIFY Natural Frequencies and Mode Shapes during Maintenance Outage (OR17)
Shutdown Experimental Modal Analysis
Post-Modification EMA

**PERFORM** Start-up Operating Modal Analysis (OMA) on FW-P-32B Monitor Natural Frequencies, Mode Shapes Verify Modification Results

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# 2005 EXPERIMENTAL MODAL ANALYSIS



**REVIEW** 2005 Experimental Modal Analysis

Data / Analysis / Methodologies / Assumptions / Conclusions

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

#### **EXPERIMENTAL MODAL ANALYSIS:**

Experimental Modal Analysis (EMA) is also known as "Impact" or "Impulse" Natural Frequency Analysis. The empirical test method is a static diagnostic performed on structures to determine their natural frequencies, damping, and mode shapes. This analysis is performed with the machinery in a non-operating condition with limited background vibration from other machinery sources.

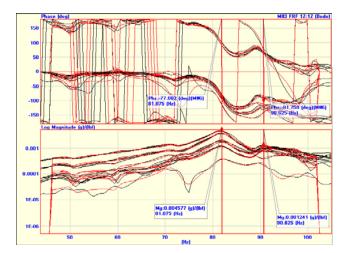
An impulse (usually a hammer blow) is applied to the structure to impart broad-band energy to the system. This energy functions to excite all of the system natural frequencies, similar to the response of a bell or tuning fork to a sharp impact. This hammer has a load cell built into the head that measures the force parted to the structure and allows this signal to be compared to the accelerometer response, via a transfer function, resulting in a measurement of the frequency response of the structure. Given enough response locations, each system natural frequency can be curve fit and its mode shape animated with specialized modal analysis software.

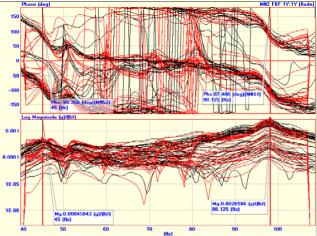
This testing was performed on the FW-P-32-B unit in each directional axis (x, y, z) separately to ensure that each mode in the operating range would be excited. The resulting data reflects mode shapes in these three dominant directions.

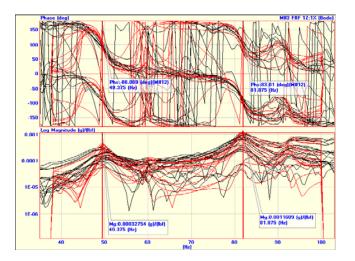
The x-axis is along the shaft centerline.

The y-axis is in the pump horizontal direction.

The z-axis is in the vertical pump direction.



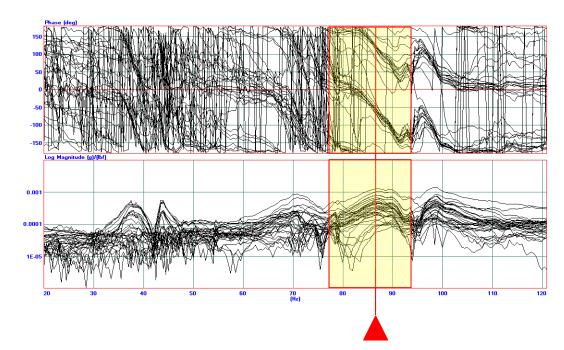




Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# 2005 EMA Analysis

- 1 37.7 Hz Rigid Body (y)
- 2 43.7 Hz Rigid Body (x, y)
- 3 69.9 Hz Vert Pitch (z)
- 4 85.2 Hz Torsion (z)
- 5 98.3 Hz Torsion (y, z)

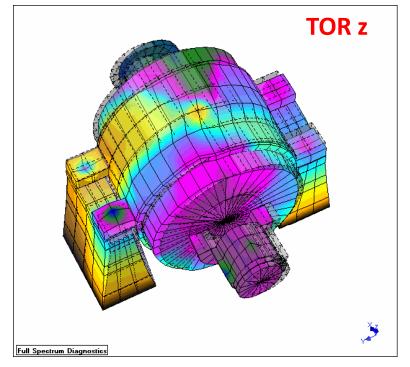


#### **EXPERIMENTAL MODAL ANALYSIS:**

The 2005 Vibration Analysis included an Experimental Modal Analysis with the FWP32B in a non-operating condition. The analysis defined several natural frequencies in the system.

A Torsional Mode of the pump was detected within 1% of the normal operating speed @ 85 Hz (5,100 RPM). Excitation of this mode by residual unbalance and alignment forces created an elevated response in the horizontal direction.

The response could be detected at each bearing but was dominant at the pump outboard bearing. The Overlaid Measurement data set below shows a Magnitude Response and Phase Angle Response (Bode Plot).





Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# 2005 EMA Analysis Modifications Margin

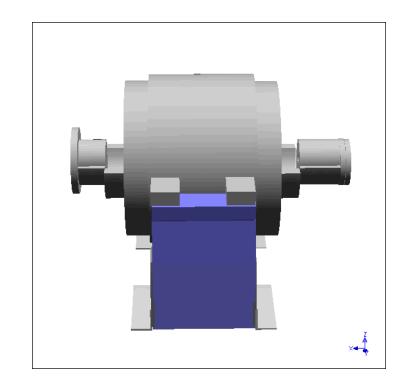
1	37.7 Hz	$\rightarrow$	44.0 Hz	-48.2 %
2	43.7 Hz	$\rightarrow$	53.3 Hz	-37.3 %
3	69.9 Hz	$\rightarrow$	<b>70.7 Hz</b>	-16.8 %
4	85.2 Hz	$\rightarrow$	92.0 Hz	+8.0%
5	98.3 Hz	$\rightarrow$	106 Hz	+24.7 %

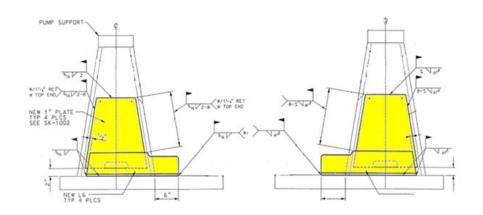


The analysis at that time predicted that a gusset design similar to the one shown here would provide sufficient stiffness to push the torsional mode out of the operating speed range and reduce the horizontal vibration to acceptable levels.

The Actual Gusset Design (shown on right) was projected to provide the frequency margins shown in the table above. Typically, a margin of +/- 10% is desired.

Note that the (Vertical Pitching ) mode at 69.9 Hz was not expected to increase due to the gusset modification, which was designed to restrict only the "torsional" response.







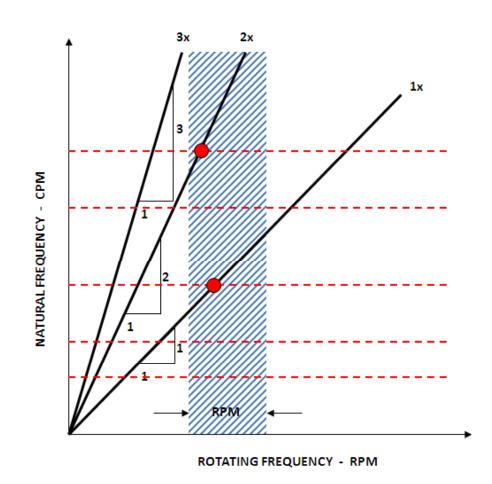
Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

#### THE CAMPBELL DIAGRAM:

A Campbell Diagram, also know as an Interference Diagram, allows the user to visually plot rotating orders of machinery versus natural frequencies in the system to determine if a coincidence situation exists, or more plainly, to determine potential resonance drivers in the system.

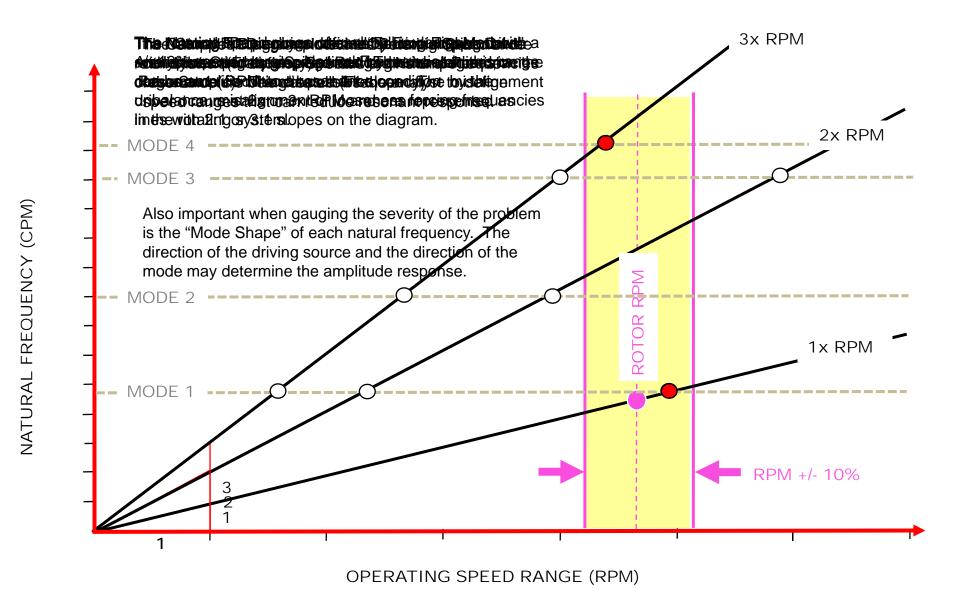
This diagram is a plot of <u>Frequency</u> (Natural Frequency) in CPM units versus <u>Frequency</u> (Rotational Speed) in RPM units. The natural frequencies of the Machine or Structures are defined from Experimental Modal testing and plotted as horizontal lines on the diagram. The operational speed frequency ranges are defined and plotted as a vertical range on the diagram. Rotating orders of running speed (1x, 2x, 3x, ....nx RPM) are plotted as sloping lines. The 1x RPM line has a 1:1 slope, the 2x RPM line has a 2:1 slope, and each higher harmonic follows the same characteristic.

Crossings of the rotating orders (1x, 2x, ...RPM) and the plotted natural frequencies within the operating speed ranges indicate possible resonance concerns that may impact the vibratory amplitudes in the system. Note that mechanical systems are designed with rotating speed margin (difference) on natural frequencies to reduce amplification due to resonance.

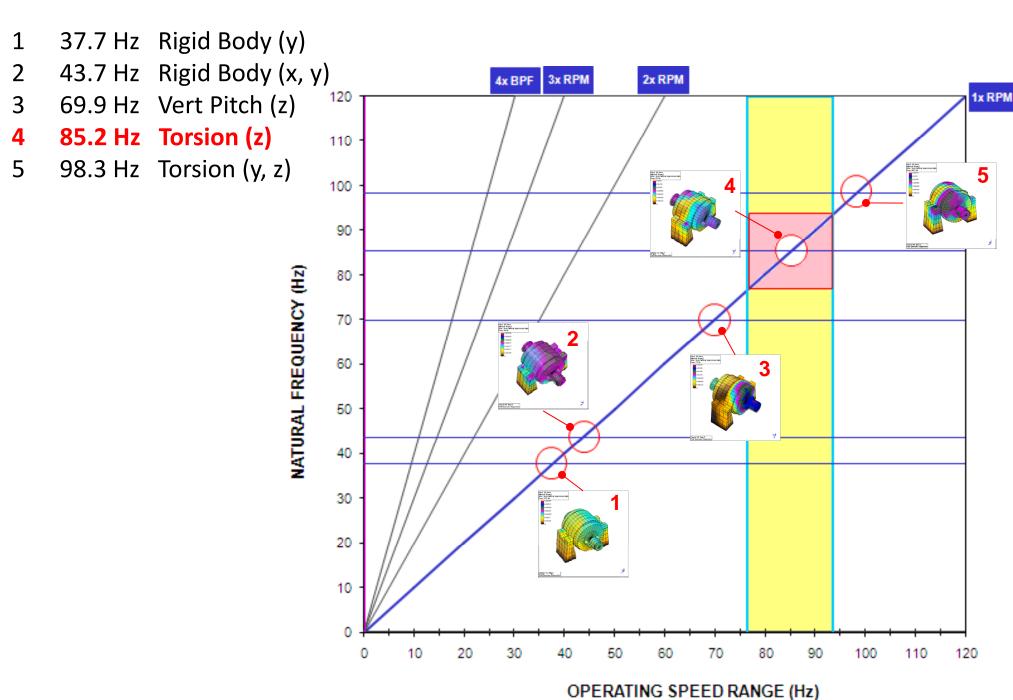


Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

### **Use Arrow Keys or Space Bar to Advance Graphics**

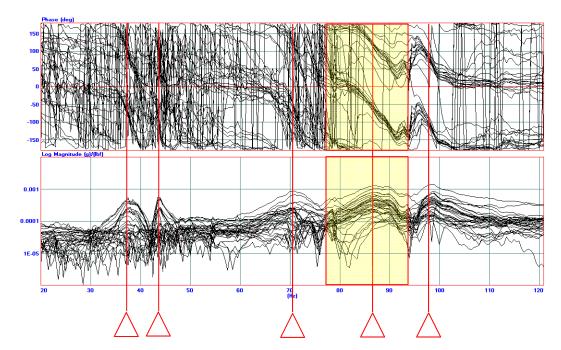


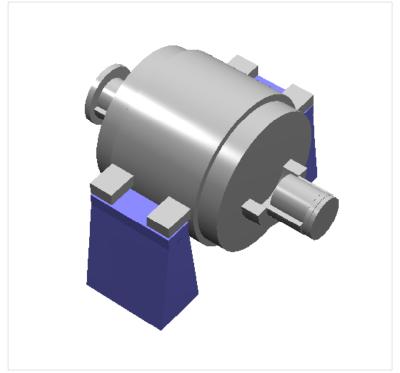
# 2005 EMA Analysis: Campbell Diagram



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

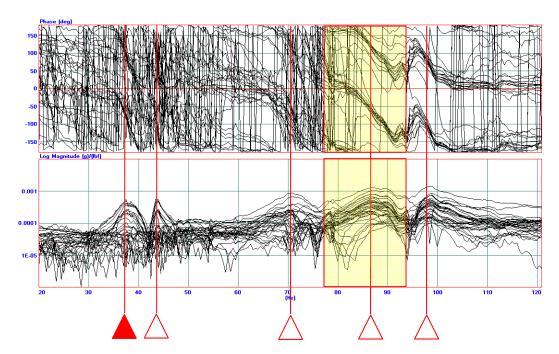
- 1 37.7 Hz Rigid Body (y)
- 2 43.7 Hz Rigid Body (x, y)
- 3 69.9 Hz Vert Pitch (z)
- 4 85.2 Hz Torsion (z)
- 5 98.3 Hz Torsion (y, z)

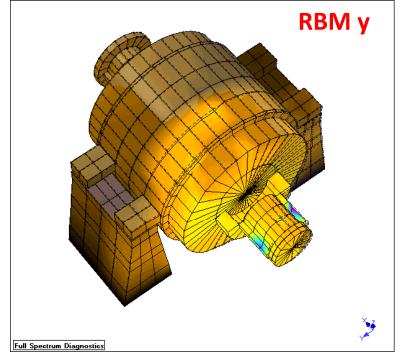




Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

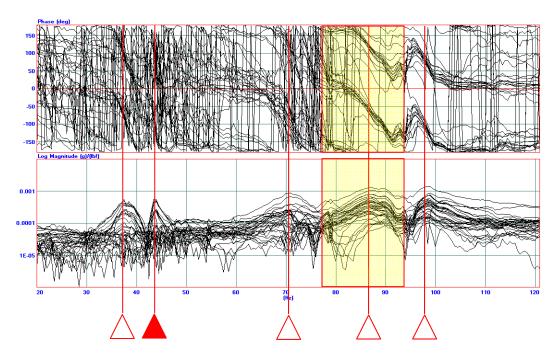
- **1** 37.7 Hz Rigid Body (y)
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- 5 98.3 Hz Torsion (y, z)

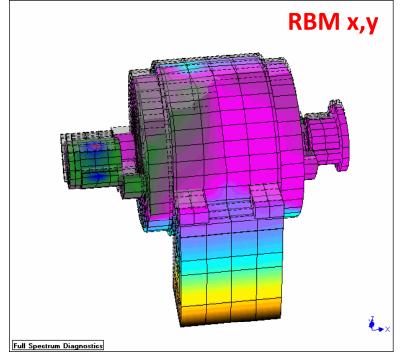




Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

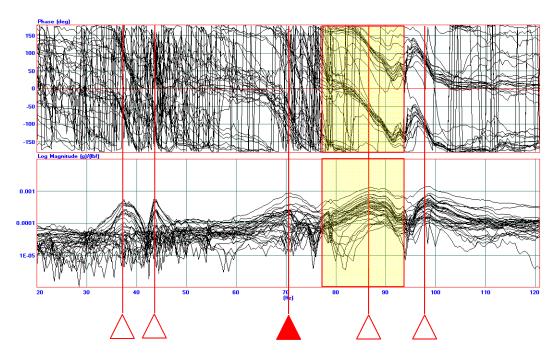
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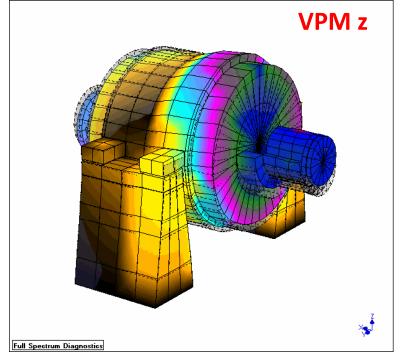




Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

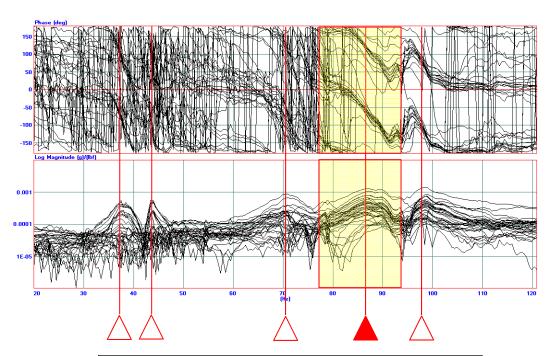
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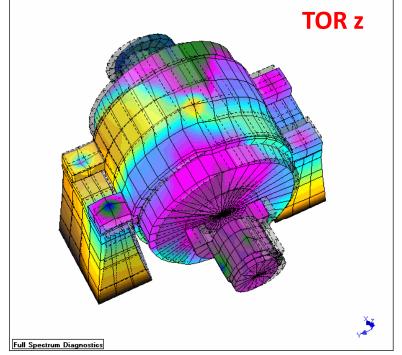




Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

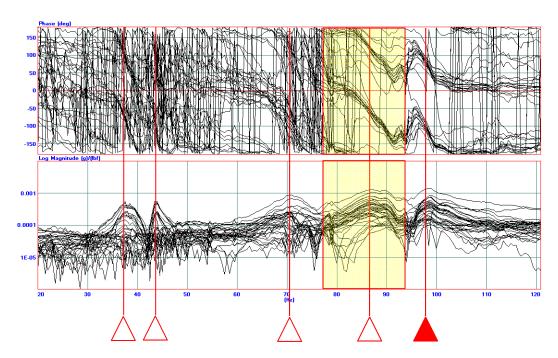
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- 3 69.9 Hz Vert Pitch (z)
- 4 85.2 Hz Torsion (z)
- 5 98.3 Hz Torsion (y, z)

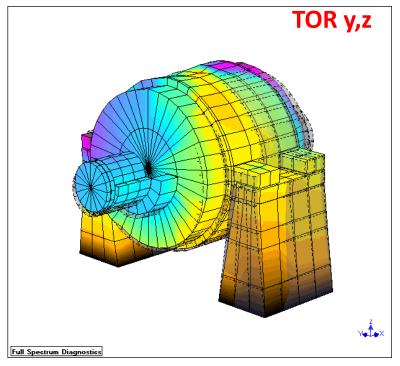




Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

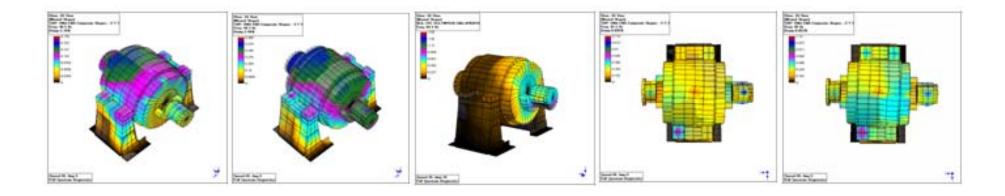
- 1 37.7 Hz Rigid Body (y)
- 2 43.7 Hz Rigid Body (x, y)
- 3 69.9 Hz Vert Pitch (z)
- 4 85.2 Hz Torsion (z)
- 5 98.3 Hz Torsion (y, z)

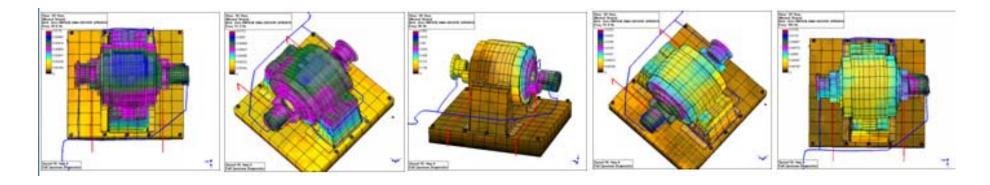




Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# **2015 OPERATING MODAL ANALYSIS**





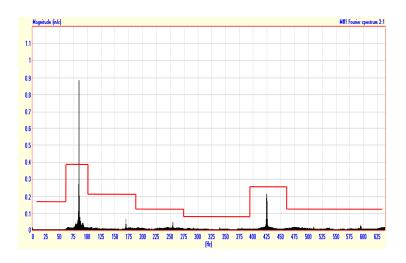
**PERFORM** Operating Modal Analysis (OMA) on FW-P-32B Determine Current Natural Frequencies & Mode Shapes

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

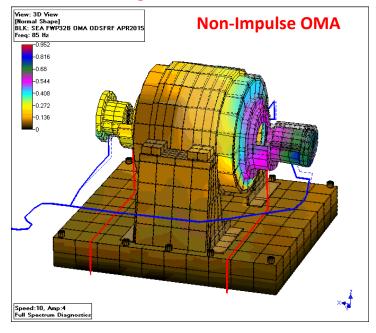
#### **OPERATING MODAL ANALYSIS:**

Operating Modal Analysis is an analysis method used to extract resonant frequencies, damping, and/or operating shapes (un-scaled mode shapes) of a structure. This method sometime called output-only modal analysis because only the response of the structure is measured. The structure might be excited using natural operating conditions or some other excitations might be applied to the structure; however, as long as the operating shapes are not scaled based on the applied force, it is called operational modal analysis.

The operating characteristics of the Feed Water Pump include a sufficient level of "random" background response from turbulent steam flow and rotor energy. If this low-level background noise is processed with significant averaging, the Modal Frequencies and Mode Shapes can be extracted from the final data. This OMA is referred to here as a "Non-Impulse Operating Modal Analysis."



### **Vertical Pitching Mode: 85 Hz**

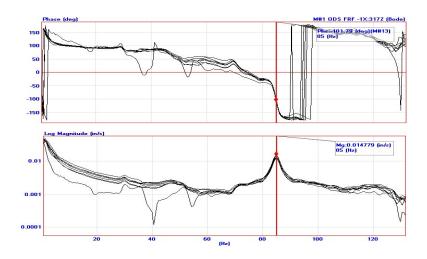


Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

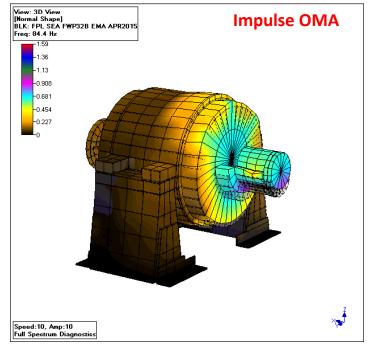
#### **OPERATING MODAL ANALYSIS:**

Operating Modal Analysis is an analysis method used to extract resonant frequencies, damping, and/or operating shapes (un-scaled mode shapes) of a structure. This method sometime called output-only modal analysis because only the response of the structure is measured. The structure might be excited using natural operating conditions or some other excitations might be applied to the structure; however, as long as the operating shapes are not scaled based on the applied force, it is called operational modal analysis.

The 2015 Vibration Analysis also included an Experimental Modal Analysis Impact Test with the FWP32B in an operating condition. This analysis is sometimes called an Operating Modal Analysis. Here it is referred to as an "Impulse Operating Modal Analysis."

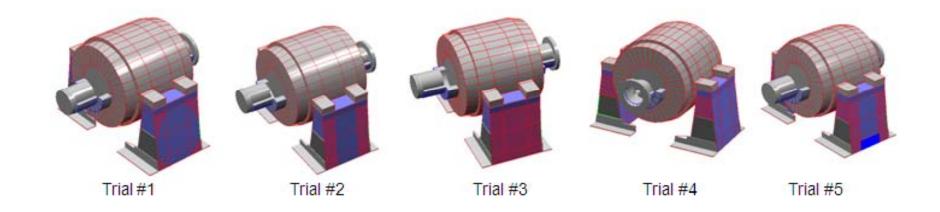


## **Vertical Pitching Mode: 85 Hz**



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# **2015 FINITE ELEMENT ANALYSIS**



CONTRUCT Finite Element Analysis (FEA) Model of Pump
Use 2005 EMA and 2015 OMA Analyses for Correlation of Modes & Mode Shapes

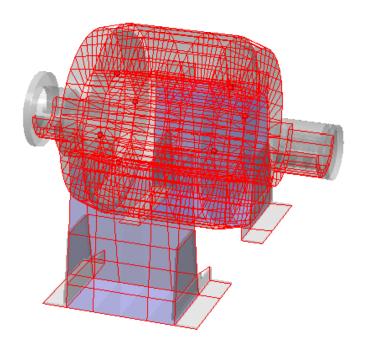
Use FEA Model to **DEFINE** Structural Modifications to Eliminate or Reduce 1x RPM Vibration Response: Stiffness / Mass / Damping / Phase Cancellation

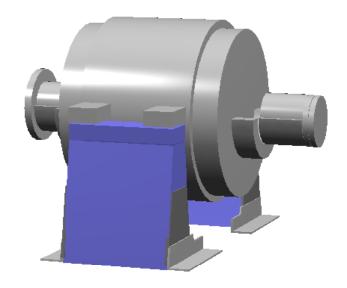
Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

#### **FINITE ELEMENT ANALYSIS:**

Finite Element Analysis, or FEA, is a numerical computer modeling method of analysis used for structural design and evaluation. This analysis involves the creation of a computer representation of a structure or machine. The representation consists of points, lines, and surfaces that define the elements, or three-dimensional volume(s) of the component under investigation.

Additional engineering material(s) properties information in the form of Mass Density (lb/in3), Modulus of Elasticity (lb/in2), Shear Modulus (lb/in2), and Poisson's Ratio are assigned to the elements. As well as artificial stiffness, added mass and flexibilities to simulate their boundary (mounted) conditions. If designed correctly, the FEA model can accurately predict the "Modes of Vibration" (natural frequencies) in the analysis structure. FEA models typically require some experimental data for empirical correlation. A common method, as in this case is a natural frequency correlation from an Experimental Modal Analysis.

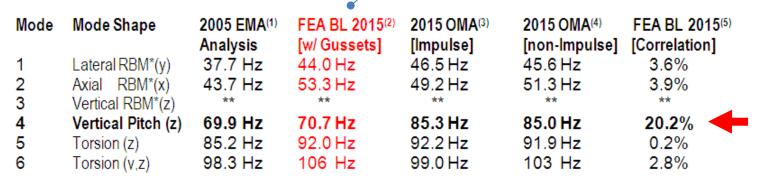




Rotating Speed: 3600 RPM Overall Alarm: 0.400 ips

#### **OPERATING MODAL ANALYSIS:**

## Pre-Shutdown Analysis & Testing



- (1) 2005 EMA Test Conditions [Cold, Coupled, Dewatered, All Piping Attached]
- (2) 2015 FEA Analysis based on 2005 Geometry and 2014 Pedestal Gussets Modification Installed
- (3) 2015 OMA Analysis (Non-Impulse) de-convolution of the ME'scope ODSFRF® measurements
- (4) 2015 OMA Analysis (Impulse) Operating EMA Analysis
- (5) FEA Baseline analysis Correlation with OMA (Impulse) and OMA (Non-Impulse) Results
- (\*) RBM denotes Rigid Body Mode. Typically, in-phase translational motion (x, y, z) without significant bending
- (\*\*) The Vertical RBM (bounce) was not detected in the 2005 EMA data & below detection in the OMA data.

Excellent Correlation with 2005 EMA analysis and Modified Pedestal Gussets introduced into the FEA model.

Except FEA model did NOT Predict the Vertical Pitching Mode to move into the operating speed range.



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## 2015 FEA Analysis



1 44.0 Hz Lateral RBM

2 53.3 Hz Axial RBM

3 \*\*\*



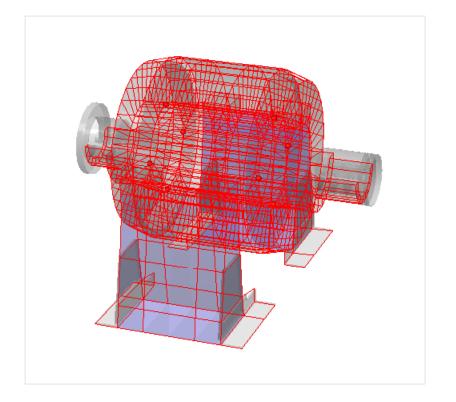
70.7 Hz Vert Pitch Mode

5 92.0 Hz Torsional Mode

106 Hz Torsional Mode

Mode	Mode Shape	2005 EMA <sup>(1)</sup> Analysis	FEA BL 2015(2) [w/ Gussets]	2015 OMA <sup>(3)</sup> [Impulse]	2015 OMA <sup>(4)</sup> [non-Impulse]	FEA BL 2015 <sup>(5)</sup> [Correlation]
1	Lateral RBM*(y)	37.7 Hz	44.0 Hz	46.5 Hz	45.6 Hz	3.6%
2	Axial RBM*(x)	43.7 Hz	53.3 Hz	49.2 Hz	51.3 Hz	3.9%
3	Vertical RBM*(z)	**	**	**	**	**
4	Vertical Pitch (z)	69.9 Hz	70.7 Hz	85.3 Hz	85.0 Hz	20.2%
5	Torsion (z)	85.2 Hz	92.0 Hz	92.2 Hz	91.9 Hz	0.2%
6	Torsion (v.z)	98.3 Hz	106 Hz	99.0 Hz	103 Hz	2.8%

- (1) 2005 EMA Test Conditions [Cold, Coupled, Dewatered, All Piping Attached]
- (2) 2015 FEA Analysis based on 2005 Geometry and 2014 Pedestal Gussets Modification Installed
- (3) 2015 OMA Analysis (Non-Impulse) de-convolution of the ME'scope ODSFRF® measurements
- (4) 2015 OMA Analysis (Impulse) Operating EMA Analysis
- (5) FEA Baseline analysis Correlation with OMA (Impulse) and OMA (Non-Impulse) Results
- (\*) RBM denotes Rigid Body Mode. Typically, in-phase translational motion (x, y, z) without significant bending
- (\*\*) The Vertical RBM (bounce) was not detected in the 2005 EMA data & below detection in the OMA data.



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## 2015 FEA Analysis



44.0 Hz Lateral RBM

53.3 Hz Axial RBM

3 \*\*\*



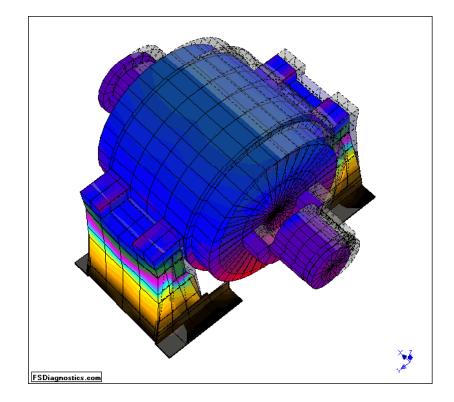
70.7 Hz Vert Pitch Mode

92.0 Hz Torsional Mode

5 106 Hz Torsional Mode

Mode	Mode Shape	2005 EMA <sup>(1)</sup> Analysis	FEA BL 2015(2) [w/ Gussets]	2015 OMA <sup>(3)</sup> [Impulse]	2015 OMA <sup>(4)</sup> [non-Impulse]	FEA BL 2015(5) [Correlation]
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- (1) 2005 EMA Test Conditions [Cold, Coupled, Dewatered, All Piping Attached]
- (2) 2015 FEA Analysis based on 2005 Geometry and 2014 Pedestal Gussets Modification Installed
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- (4) 2015 OMA Analysis (Impulse) Operating EMA Analysis
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Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## 2015 FEA Analysis



44.0 Hz Lateral RBM

53.3 Hz Axial RBM

3 \*\*\*

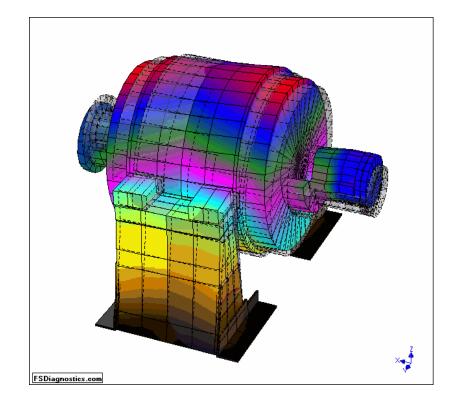
70.7 Hz Vert Pitch Mode

92.0 Hz Torsional Mode

106 Hz Torsional Mode

Mode	Mode Shape	2005 EMA <sup>(1)</sup> Analysis	FEA BL 2015(2) [w/ Gussets]	2015 OMA <sup>(3)</sup> [Impulse]	2015 OMA <sup>(4)</sup> [non-Impulse]	FEA BL 2015(5) [Correlation]
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- (1) 2005 EMA Test Conditions [Cold, Coupled, Dewatered, All Piping Attached]
- (2) 2015 FEA Analysis based on 2005 Geometry and 2014 Pedestal Gussets Modification Installed
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- (\*\*) The Vertical RBM (bounce) was not detected in the 2005 EMA data & below detection in the OMA data.



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## 2015 FEA Analysis



44.0 Hz Lateral RBM

53.3 Hz Axial RBM

**\*\***\*

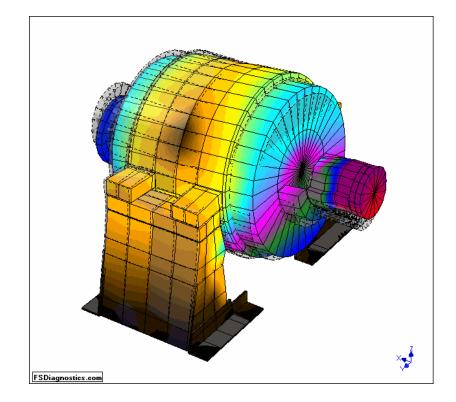
70.7 Hz Vert Pitch Mode

92.0 Hz Torsional Mode

106 Hz Torsional Mode

Mode	Mode Shape	2005 EMA <sup>(1)</sup> Analysis	FEA BL 2015(2) [w/ Gussets]	2015 OMA <sup>(3)</sup> [Impulse]	2015 OMA <sup>(4)</sup> [non-Impulse]	FEA BL 2015(5) [Correlation]
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6	Torsion (y,z)	98.3 Hz	106 Hz	99.0 Hz	103 Hz	2.8%

- (1) 2005 EMA Test Conditions [Cold, Coupled, Dewatered, All Piping Attached]
- (2) 2015 FEA Analysis based on 2005 Geometry and 2014 Pedestal Gussets Modification Installed
- (3) 2015 OMA Analysis (Non-Impulse) de-convolution of the ME'scope ODSFRF® measurements
- (4) 2015 OMA Analysis (Impulse) Operating EMA Analysis
- (5) FEA Baseline analysis Correlation with OMA (Impulse) and OMA (Non-Impulse) Results
- (\*) RBM denotes Rigid Body Mode. Typically, in-phase translational motion (x, y, z) without significant bending
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Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## 2015 FEA Analysis



44.0 Hz Lateral RBM

53.3 Hz Axial RBM

\*\*\*



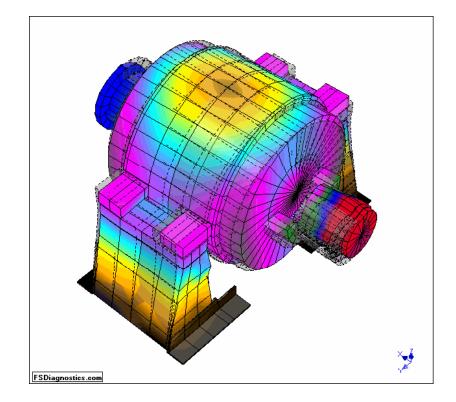
70.7 Hz Vert Pitch Mode

92.0 Hz Torsional Mode

106 Hz Torsional Mode

Mode	Mode Shape	2005 EMA <sup>(1)</sup> Analysis	FEA BL 2015(2) [w/ Gussets]	2015 OMA <sup>(3)</sup> [Impulse]	2015 OMA <sup>(4)</sup> [non-Impulse]	FEA BL 2015(5) [Correlation]
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5	Torsion (z)	85.2 Hz	92.0 Hz	92.2 Hz	91.9 Hz	0.2%
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- (1) 2005 EMA Test Conditions [Cold, Coupled, Dewatered, All Piping Attached]
- (2) 2015 FEA Analysis based on 2005 Geometry and 2014 Pedestal Gussets Modification Installed
- (3) 2015 OMA Analysis (Non-Impulse) de-convolution of the ME'scope ODSFRF® measurements
- (4) 2015 OMA Analysis (Impulse) Operating EMA Analysis
- (5) FEA Baseline analysis Correlation with OMA (Impulse) and OMA (Non-Impulse) Results
- (\*) RBM denotes Rigid Body Mode. Typically, in-phase translational motion (x, y, z) without significant bending
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Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## 2015 FEA Analysis



44.0 Hz Lateral RBM

2 53.3 Hz Axial RBM

**2** \*\*\*



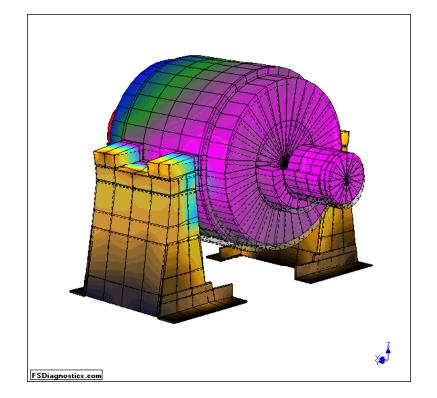
70.7 Hz Vert Pitch Mode

92.0 Hz Torsional Mode

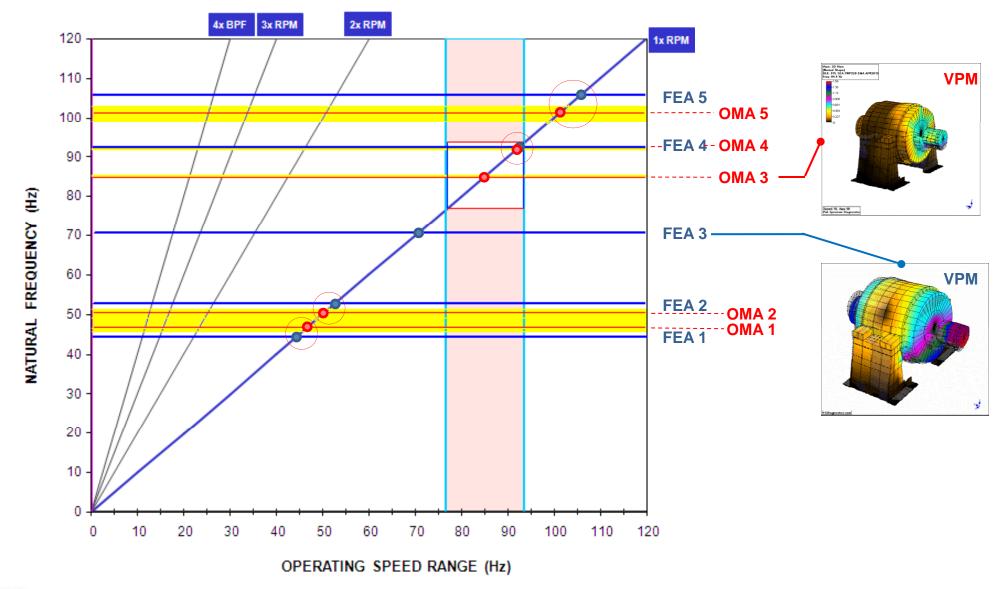
106 Hz Torsional Mode

Mode	Mode Shape	2005 EMA <sup>(1)</sup> Analysis	FEA BL 2015(2) [w/ Gussets]	2015 OMA <sup>(3)</sup> [Impulse]	2015 OMA <sup>(4)</sup> [non-Impulse]	FEA BL 2015(5) [Correlation]
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2	Axial RBM*(x)	43.7 Hz	53.3 Hz	49.2 Hz	51.3 Hz	3.9%
3	Vertical RBM*(z)	**	**	**	**	**
4	Vertical Pitch (z)	69.9 Hz	70.7 Hz	85.3 Hz	85.0 Hz	20.2%
5	Torsion (z)	85.2 Hz	92.0 Hz	92.2 Hz	91.9 Hz	0.2%
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- (1) 2005 EMA Test Conditions [Cold, Coupled, Dewatered, All Piping Attached]
- (2) 2015 FEA Analysis based on 2005 Geometry and 2014 Pedestal Gussets Modification Installed
- (3) 2015 OMA Analysis (Non-Impulse) de-convolution of the ME'scope ODSFRF® measurements
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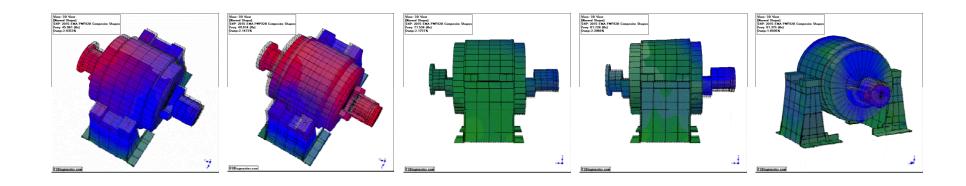


Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## **2015 EXPERIMENTAL MODAL ANALYSIS**

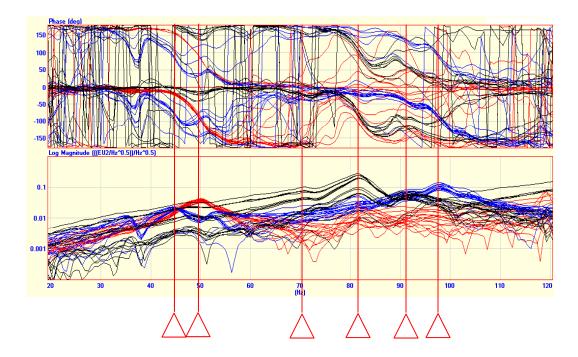


**VERIFY** Natural Frequencies and Mode Shapes during Maintenance Outage (OR17) Shutdown Experimental Modal Analysis

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# 2015 EMA Analysis Pre-Modifications

- 1 47.4 Hz Lateral RBM
- 2 51.2 Hz Axial RBM
- 3 72.9 Hz Vertical RBM
- 4 83.7 Hz Vert Pitch Mode
- 5 97.0 Hz Torsional Mode
- 6 100 Hz Torsional Mode



Mode	Mode Shape	2015 EMA(1) [@ SD (hot)]	2015 EMA <sup>(2)</sup> [@ SD (cold)]	2015 EMA BL <sup>(3)</sup> [@SD Dewater]
1	Lateral RBM* (y)	46.0 Hz	46.7 Hz	47.4 Hz
2	Axial RBM* (x)	49.8 Hz	49.6 Hz	51.2 Hz
3	Vertical RBM* (z)	71.5 Hz	71.4 Hz	72.9 Hz
4	Vertical Pitch (z)	81.7 Hz	81.7 Hz	83.7 Hz
5	Torsion (z)	91.4 Hz	90.6 Hz	97.0 Hz
6	Torsion (y, z)	97.0 Hz	97.5 Hz	100 Hz

- (1) EMA Test Oct 1, 2015 Conditions [Hot, Coupled, Full of Water, All Piping Attached]
- (2) EMA Test Oct 2, 2015 Conditions [Cold, Coupled, Full of Water, All Piping Attached]
- (3) EMA Baseline Test Oct 7, 2015 Conditions [Cold, Un-Coupled, Dewatered

# 2015 EMA Analysis Pre-modifications

1 47.4 Hz

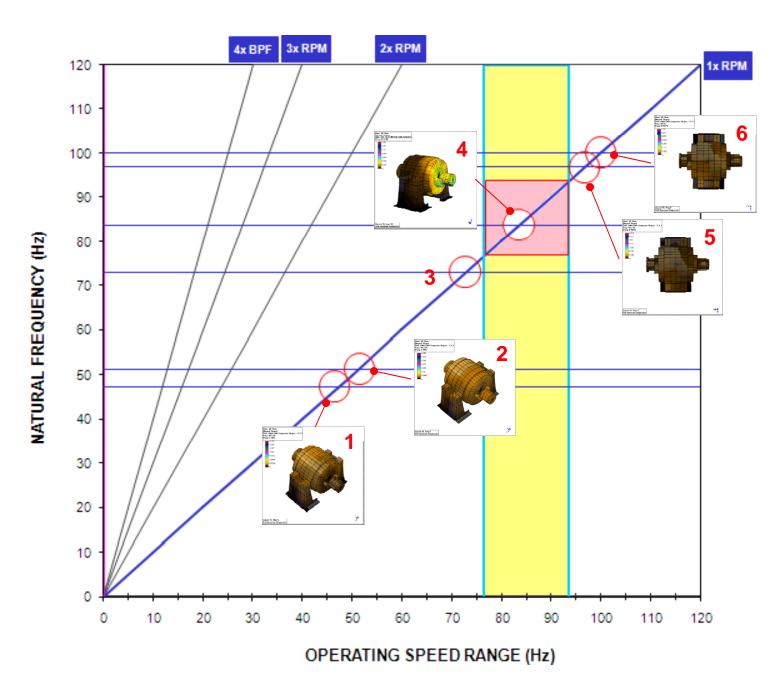
2 51.2 Hz

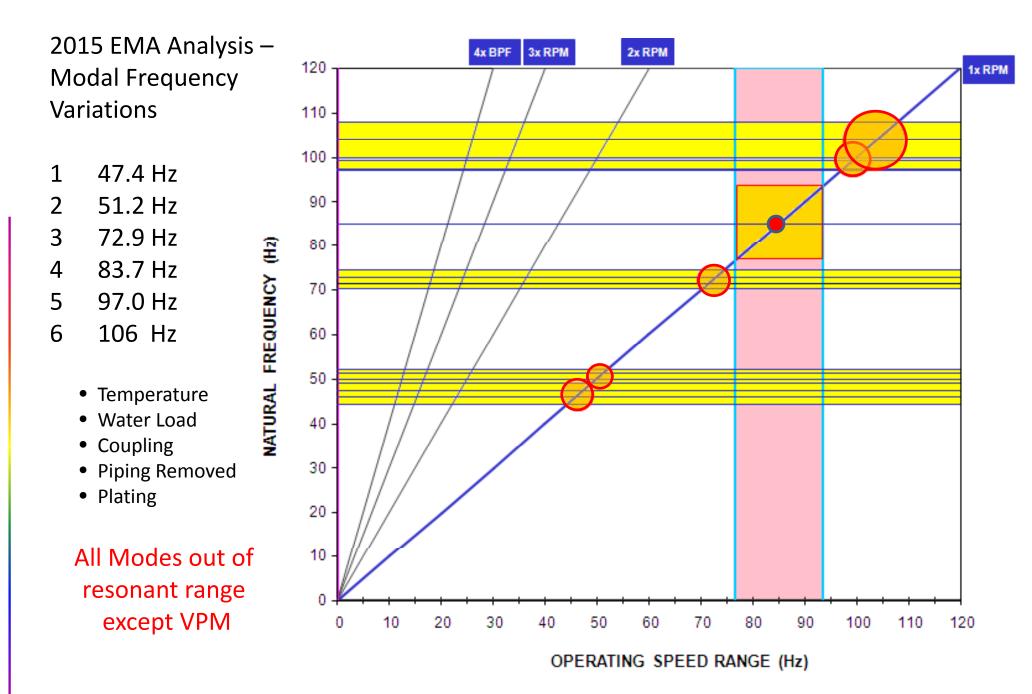
3 72.9 Hz

4 83.7 Hz

5 97.0 Hz

6 106 Hz





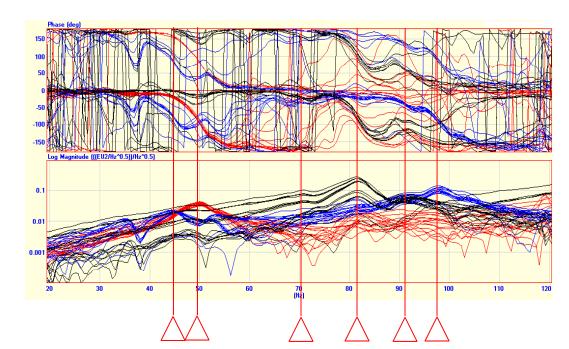
**Rotating Speed: 5100 RPM** Overall Alarm: 0.400 ips

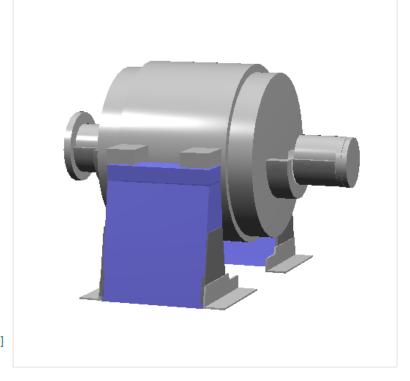
## 2015 EMA Analysis **Pre-Modifications**

- 47.4 Hz Lateral RBM
- 51.2 Hz Axial RBM
- 72.9 Hz Vertical RBM
- 83.7 Hz Vert Pitch Mode
- 97.0 Hz Torsional Mode
- 100 Hz Torsional Mode

Mode	Mode Shape	2015 EMA(1)	2015 EMA(2)	2015 EMA BL(3)
		[@ SD (hot)]	[@ SD (cold)]	[@SD Dewater]
1	Lateral RBM* (y)	46.0 Hz	46.7 Hz	47.4 Hz
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3	Vertical RBM* (z)	71.5 Hz	71.4 Hz	72.9 Hz
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5	Torsion (z)	91.4 Hz	90.6 Hz	97.0 Hz
6	Torsion (y, z)	97.0 Hz	97.5 Hz	100 Hz

- (1) EMA Test Oct 1, 2015 Conditions [Hot, Coupled, Full of Water, All Piping Attached]
   (2) EMA Test Oct 2, 2015 Conditions [Cold, Coupled, Full of Water, All Piping Attached]
- (3) EMA Baseline Test Oct 7, 2015 Conditions [Cold, Un-Coupled, Dewatered, Small Line Piping Removed]







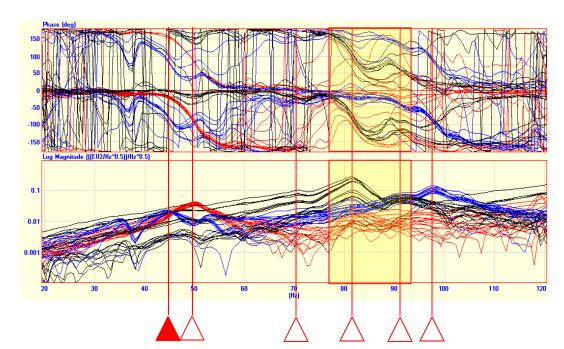
**Rotating Speed: 5100 RPM** Overall Alarm: 0.400 ips

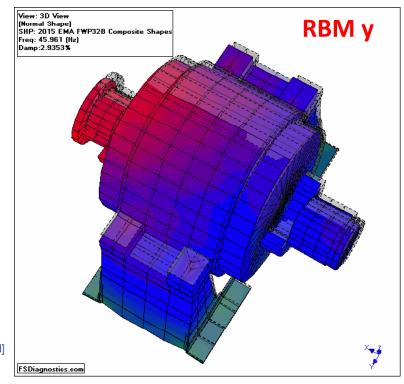
## 2015 EMA Analysis **Pre-Modifications**

- 47.4 Hz Lateral RBM
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- (1) EMA Test Oct 1, 2015 Conditions [Hot, Coupled, Full of Water, All Piping Attached]
   (2) EMA Test Oct 2, 2015 Conditions [Cold, Coupled, Full of Water, All Piping Attached]
- (3) EMA Baseline Test Oct 7, 2015 Conditions [Cold, Un-Coupled, Dewatered, Small Line Piping Removed]







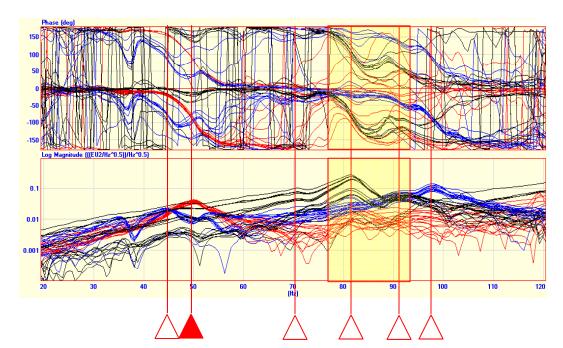
**Rotating Speed: 5100 RPM** Overall Alarm: 0.400 ips

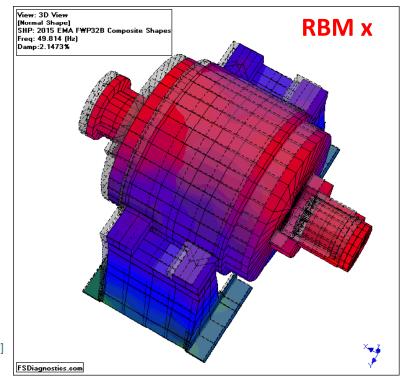
## 2015 EMA Analysis **Pre-Modifications**

- 47.4 Hz Lateral RBM
- 51.2 Hz Axial RBM
- 72.9 Hz Vertical RBM
- 83.7 Hz Vert Pitch Mode
- 97.0 Hz Torsional Mode
- 100 Hz Torsional Mode

Mode	Mode Shape	2015 EMA(1)	2015 EMA(2)	2015 EMA BL(3)
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6	Torsion (y, z)	97.0 Hz	97.5 Hz	100 Hz

- (1) EMA Test Oct 1, 2015 Conditions [Hot, Coupled, Full of Water, All Piping Attached]
   (2) EMA Test Oct 2, 2015 Conditions [Cold, Coupled, Full of Water, All Piping Attached]
- (3) EMA Baseline Test Oct 7, 2015 Conditions [Cold, Un-Coupled, Dewatered, Small Line Piping Removed]







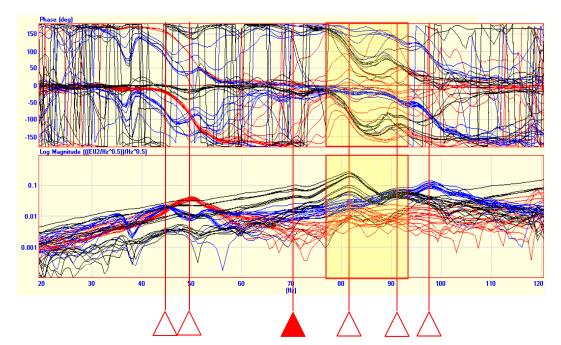
**Rotating Speed: 5100 RPM** Overall Alarm: 0.400 ips

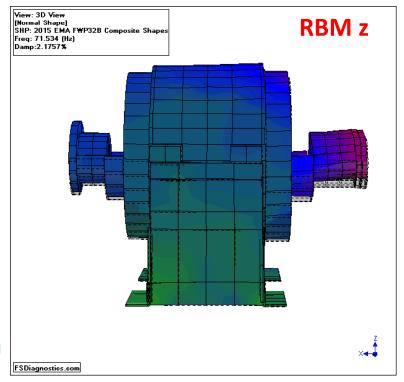
## 2015 EMA Analysis **Pre-Modifications**

- 47.4 Hz Lateral RBM
- 51.2 Hz Axial RBM
- 72.9 Hz Vertical RBM
- 83.7 Hz Vert Pitch Mode
- 97.0 Hz Torsional Mode
- 100 Hz Torsional Mode

Mode	Mode Shape	2015 EMA(1)	2015 EMA(2)	2015 EMA BL(3)
		[@ SD (hot)]	[@ SD (cold)]	[@SD Dewater]
1	Lateral RBM* (y)	46.0 Hz	46.7 Hz	47.4 Hz
2	Axial RBM* (x)	49.8 Hz	49.6 Hz	51.2 Hz
3	Vertical RBM* (z)	71.5 Hz	71.4 Hz	72.9 Hz
4	Vertical Pitch (z)	81.7 Hz	81.7 Hz	83.7 Hz
5	Torsion (z)	91.4 Hz	90.6 Hz	97.0 Hz
6	Torsion (y, z)	97.0 Hz	97.5 Hz	100 Hz

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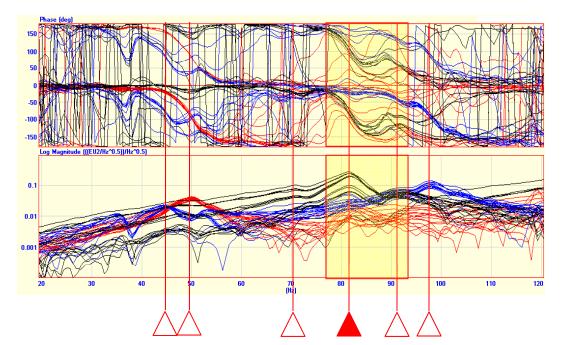
**Rotating Speed: 5100 RPM** Overall Alarm: 0.400 ips

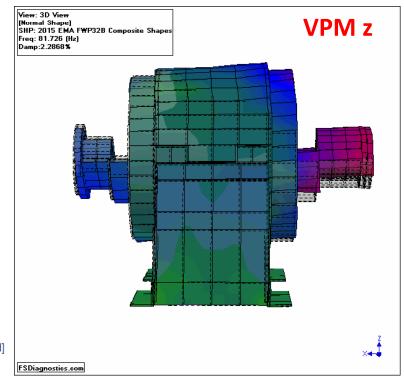
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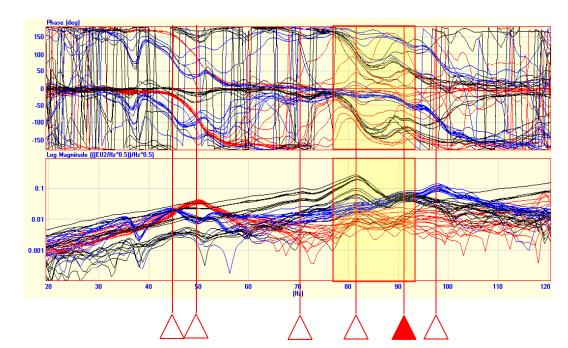
**Rotating Speed: 5100 RPM** Overall Alarm: 0.400 ips

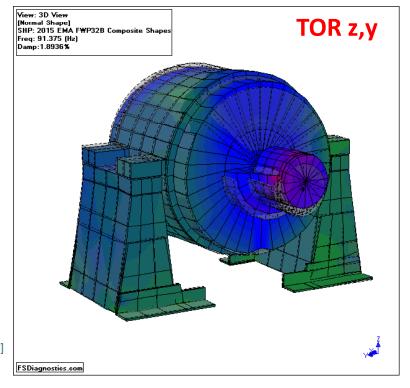
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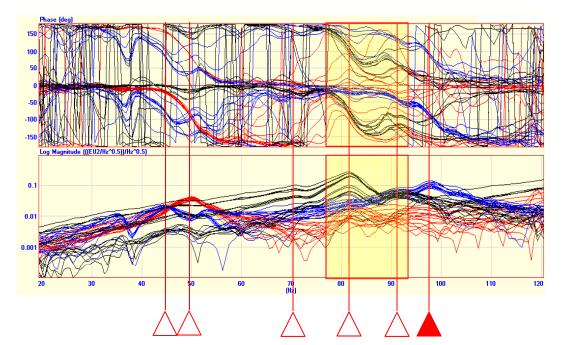
**Rotating Speed: 5100 RPM** Overall Alarm: 0.400 ips

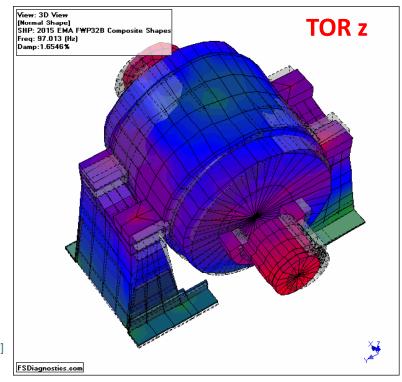
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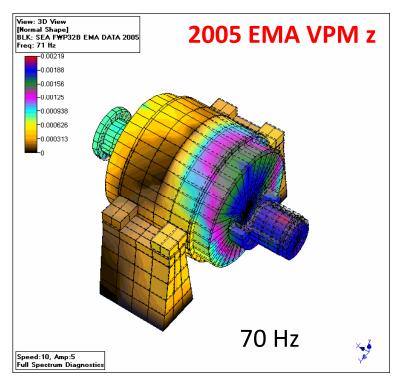
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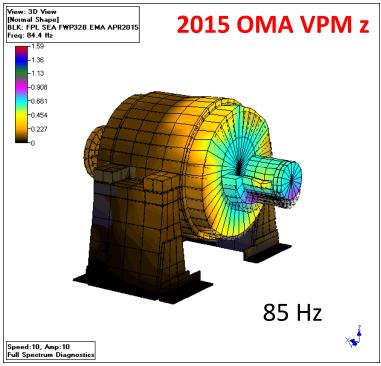


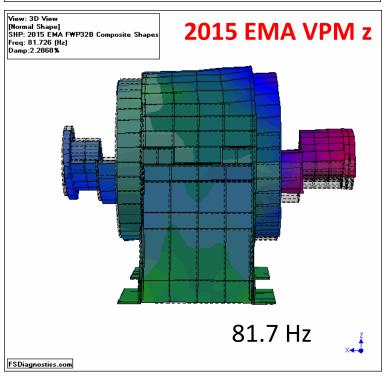




Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

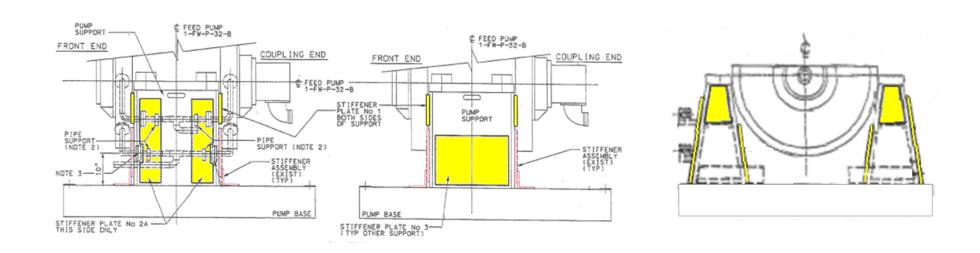






Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# **2015 STRUCTURAL MODIFICATIONS**



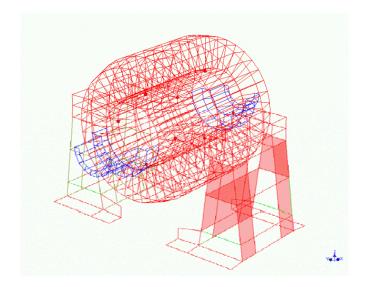
VERIFY Natural Frequencies and Mode Shapes during Maintenance Outage (OR17)

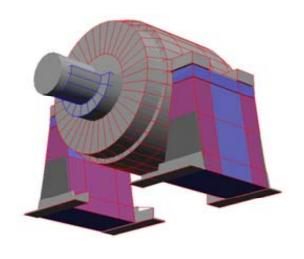
Shutdown Experimental Modal Analysis

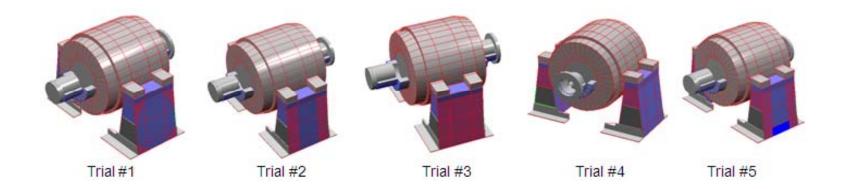
Post-Modification EMA

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# **2015 STRUCTURAL MODIFICATIONS**





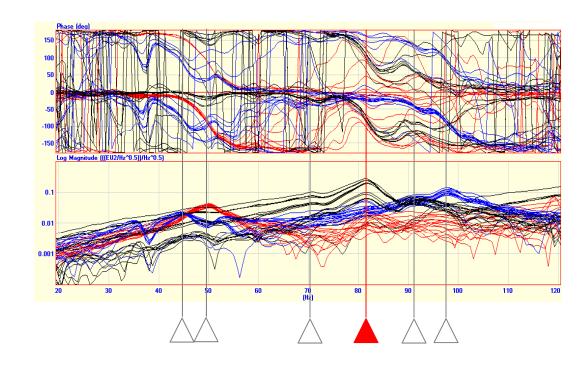


Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# 2015 EMA Analysis Post Plating Modifications

- 1 49.1 Hz
- 2 52.1 Hz
- 3 74.5 Hz
- 4 86.9 Hz [Plating Δ 3.2 Hz]
- 5 97.4 Hz
- 6 104 Hz





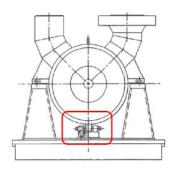
## EMA Correlated FEA / SDM Plating (stiffening) Analysis

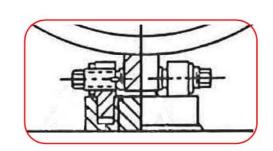
Mode	Mode Shape	SDM Plating <sup>(3)</sup> [EMA BL Projected]	2015 EMA <sup>(4)</sup> [Final Plating]
1	Lateral RBM* (y)	58.9 Hz	49.1 Hz
2	Axial RBM* (x)	74.8 Hz	52.1 Hz
3	Vertical RBM* (z)	75.3 Hz	74.5 Hz
4	Vertical Pitch (z)	96.8 Hz	86.9 Hz
5	Torsion (z)	92.3 Hz	97.4 Hz
6	Torsion (y, z)	113 Hz	104 Hz

- (3) SDM Plating Projection based on Baseline EMA analysis
- (4) EMA Test Final Plating Complete Oct 16, 2015

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## **2015 ALIGNMENT GUIDE BARREL LUGS**



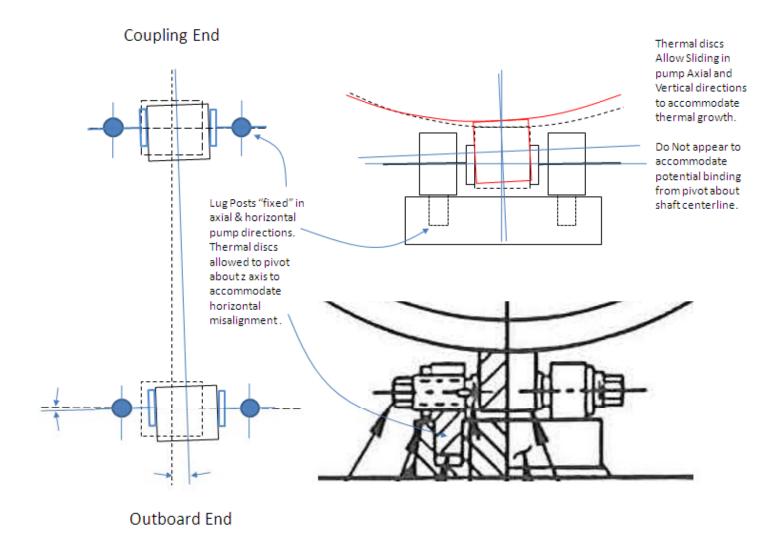




DIGGING DEEPER Searching for the Discrepancies in the Analysis vs. Data Potential External Forces on Pump Piping / Foundation / Alignment Guide Lugs

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

## Attained Approval to "Un-Lock" the Alignment Guide Barrel Lugs

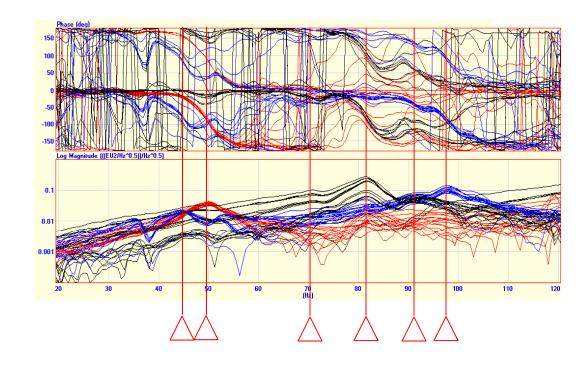


Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# 2015 EMA Analysis Post-Modifications Lugs Released

- 1 44.1 Hz
- 2 50.6 Hz
- 3 70.6 Hz
- 4 76.8 Hz [Δf: -11.6%]
- 5 102 Hz
- 6 109 Hz

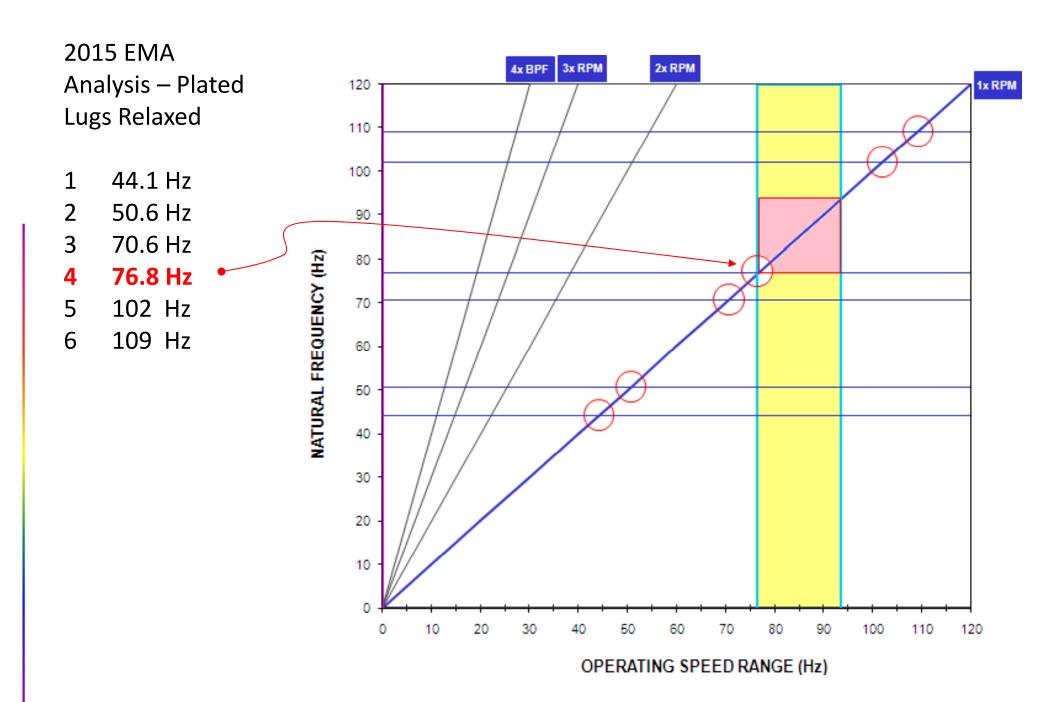




## EMA Correlated FEA / SDM Plating (stiffening) Analysis

Mode	Mode Shape	SDM Plating <sup>(3)</sup> [EMA BL Projected]	2015 EMA <sup>(4)</sup> [Final Plating]	2015 EMA <sup>(5)</sup> [Lugs Released]
1	Lateral RBM* (y)	58.9 Hz	49.1 Hz	44.1 Hz
2	Axial RBM* (x)	74.8 Hz	52.1 Hz	50.6 Hz
3	Vertical RBM* (z)	75.3 Hz	74.5 Hz	70.6 Hz
4	Vertical Pitch (z)	96.8 Hz	86.9 Hz	76.8 Hz
5	Torsion (z)	92.3 Hz	97.4 Hz	102 Hz
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- (3) SDM Plating Projection based on Baseline EMA analysis [Cold, Uncoupled, Dewatered
- (4) EMA Test Final Plating Complete Oct 16, 2015
- (5) EMA Test Final Plating Complete, Alignment Guide Lugs Released Oct 18, 2015.



**Rotating Speed: 5100 RPM** Overall Alarm: 0.400 ips

## Pump Alignment Centering Lug Sensitivity Testing

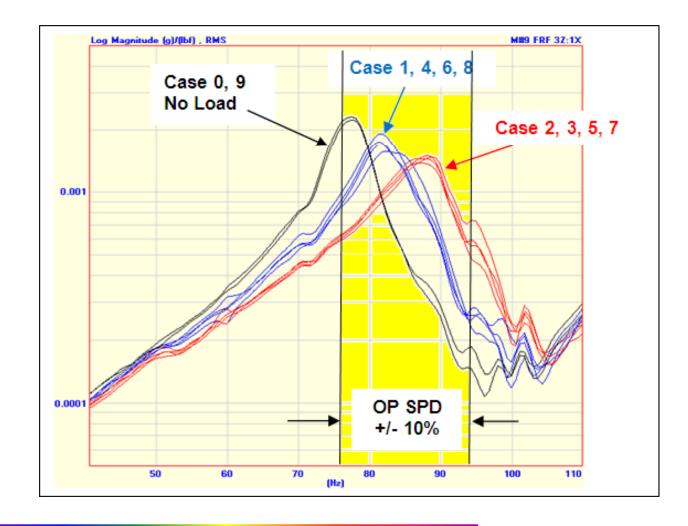
Test Case	Load <sup>(1)</sup> Configuration	Natural Frequency	Shift Frequency	Alignment <sup>(2)</sup> Effect	Matrix Diagi	Load ram	East
0	Unloaded	77.7 Hz		0.0000		$\Box$	
1	SE Lug	84.1 Hz	+6.40 Hz	0.0007 W		•	
2	SE/NE Lugs	89.2 Hz	+11.5 Hz	0.0010 W	•	r i	West
3	NW/SE Lugs	89.5 Hz	+11.8 Hz	0.0007 W	•		
4	NW Lug	81.6 Hz	+3.90 Hz	0.0000	•		
5	NW/SW Lug	89.4 Hz	+11.7 Hz	0.0004 E	•	<b>—</b>	· —
6	SW Lug	82.0 Hz	+4.30 Hz	0.0004 E		ķ.	
7	NE/SW Lugs	89.1 Hz	+11.4 Hz	0.0000	<u> </u>		
8	NE Lug	81.4 Hz	+3.70 Hz	0.0003 W			\
9	Unloaded	77.9 Hz	+0.20 Hz	0.0000			-01-11-10-
10	All Hand Tight	80.0 Hz	+2.30 Hz	0.0000	<b>‡</b>	*	PA A

- (1) All Loadings were Hand Tight + 1 Flat Engagement(2) Alignment was monitored Real Time. Dimensions are inches

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

Sensitivity Testing proved that the Lug Loading induced a significant change in the response of the Vertical Pitching Mode (VPM<sub>Z</sub>).

Testing indicated that the VPM<sub>7</sub> natural frequency was potentially "tunable".

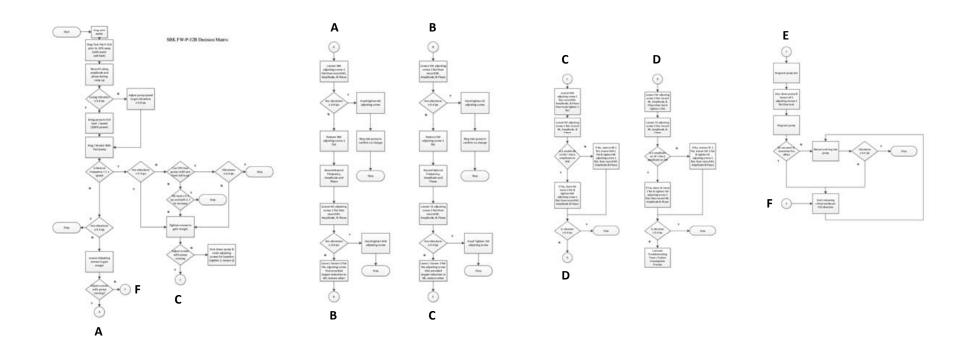


Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

# **2015 UNIT START-UP**

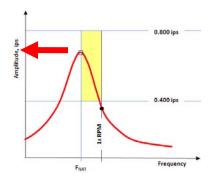
**PERFORM** Start-up Operating Modal Analysis (OMA) on FW-P-32B Monitor Natural Frequencies, Mode Shapes Verify Modification Results

**Added Potential "Tuning" Decision Matrix** 



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

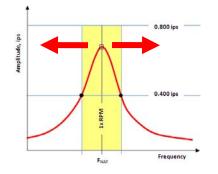
# **Decision Matrix Summary**



Natural Frequency Below 1x RPM

**Response Exceeds Limits** 

Unload Lugs to Reduce Stiffness / Attain Margin

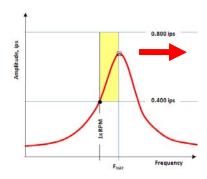


Natural Frequency

@ 1x RPM

**Response Exceeds Limits** 

Load or Unload Lugs as required to Attain Margin



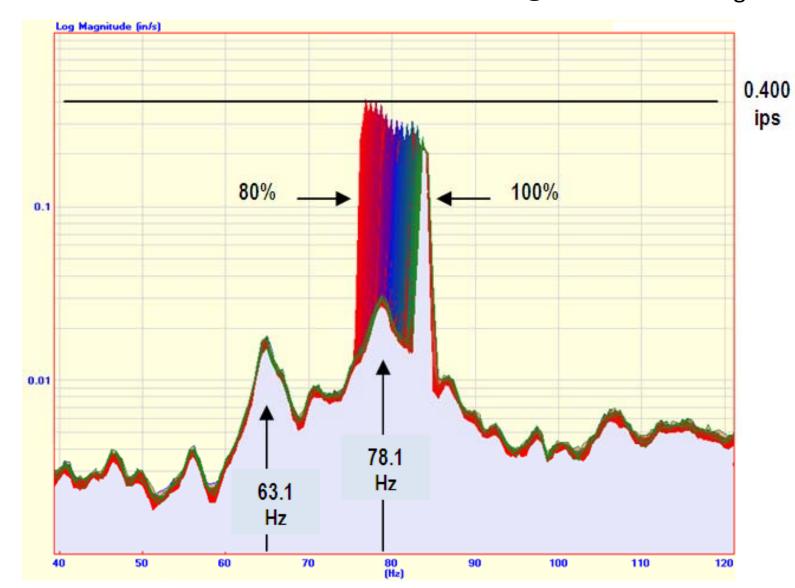
Natural Frequency
Above 1x RPM

**Response Exceeds Limits** 

Load Lugs to Increase Stiffness / Attain Margin

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

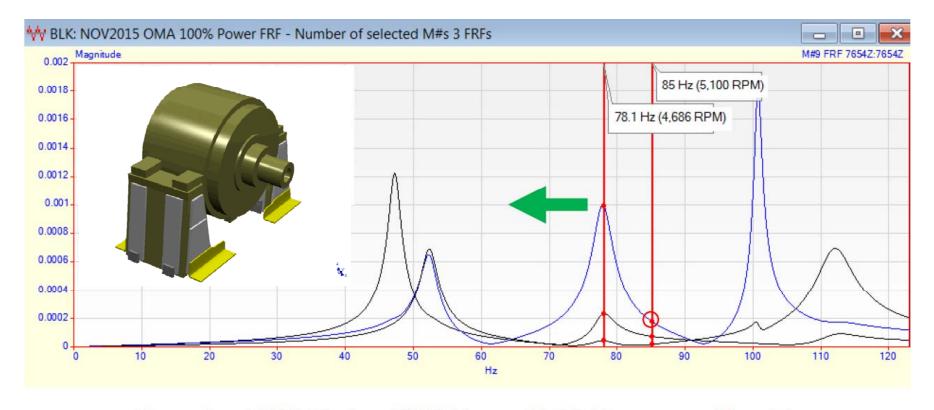
100% Power – Acceptable Response60% Reduction in Vertical Vibration @ Outboard Bearing



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips

DATE	CONDITION	NAT FREQ	AMPL FACTOR
15APR15	100% OMA (Impulse)	5,118	0.700 ips
15APR15	100% OMA (Non-Impulse)	5,100	0.700 ips
010CT15	S/D Hot EMA S/D Cold EMA S/D Cold, Uncap, Dwater S/D Plating Complete S/D AG Lugs Relaxed	81.7 Hz	Q = 21.4
020CT15		81.7 Hz	Q = 21.4
070CT15		83.7 Hz	Q = 20.1 Damping Increase Due to Plating
160CT15		86.9 Hz	Q = 17.7
180CT15		76.8 Hz	Q = 24.0
07NOV15	S/D Lug Sens (0,9)	77.5 Hz	Q = 17.1 Damping Lugs Disengaged Q = 10.9 Q = 11.5
07NOV15	S/D Lug Sens (2,3,5,7)	82.4 Hz	
07NOV15	S/D Lug Sens (1,4,6,8)	87.9 Hz	
08NOV15	S/D Final Alignment	80.0 Hz	Q = 13.9 Damping Lugs "Hand Tight" ***
15NOV15	S/D Filler w/Water	78.0 Hz	
DATE	CONDITION	1x RPM	AMP (ips)
16NOV15 17NOV15 17NOV15 17NOV15 17NOV15 17NOV15 17NOV15 17NOV15	Start 80.0% Power Start 89.7% Power Start 91.7% Power Start 92.2% Power Start 93.7% Power Start 93.7% Power Start 93.7% Power Start 93.7% Power Start 93.9% Power	4,536 4,730 4,779 4,827 4,867 4,870 4,866 4,870	0.278 ips 0.380 ips 0.310 ips 0.270 ips 0.280 ips 0.280 ips 0.280 ips 0.280 ips
17NOV15	Start 100% Power	5,037	0.225 ips •

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips



November 2015 Startup 100% Power (8.1% Resonance Margin)

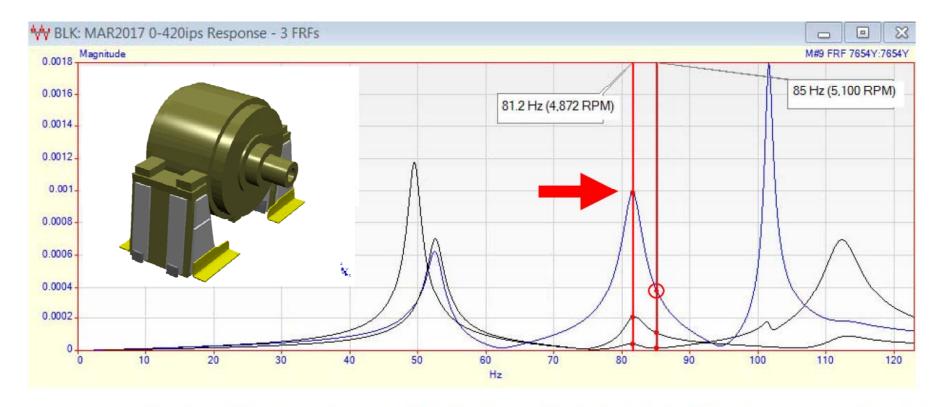
Natural Frequency: 78.1 Hz (4,686 CPM) Operating Speed: 85.0 Hz (5,100 RPM)

1x RPM Amplitude at 85 Hz: 0.225 ips

**Resonance Margin: 8.1%** 



Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips



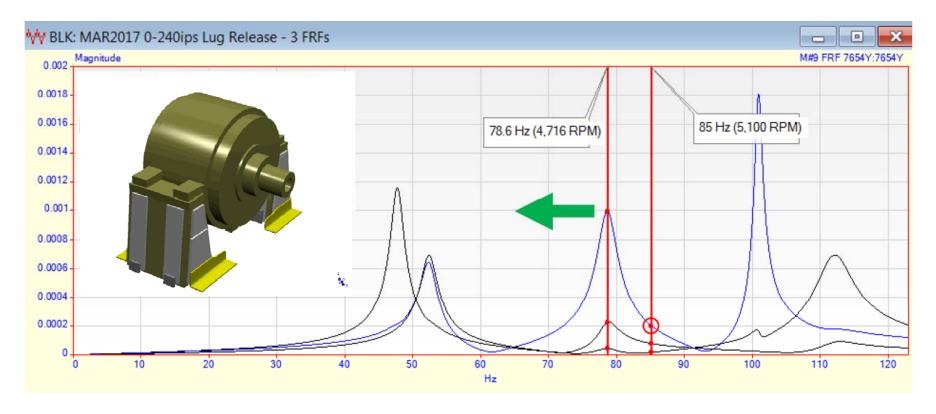
March 2017 Operating @ 100% Power (0.420 ips) (4.5% Resonance Margin)

Natural Frequency: 81.2 Hz (4,872 CPM) Operating Speed: 85.0 Hz (5,100 RPM)

1x RPM Amplitude at 85 Hz: 0.420 ips

**Resonance Margin: 4.5%** 

Rotating Speed: 5100 RPM Overall Alarm: 0.400 ips



March 2017 100% Pwr Lugs Released (0.240 ips) (7.5% Resonance Margin)

Natural Frequency: 78.6 Hz (4,716 CPM) Operating Speed: 85.0 Hz (5,100 RPM)

1x RPM Amplitude at 85 Hz: 0.240 ips

**Resonance Margin: 7.5%** 

# Vibrant Technology's ME'scope

- Operating Deflection Shape (ODS) Analysis
- Experimental Modal Analysis (EMA)
- Operating Modal Analysis (OMA)
- Structural Dynamic Modification (SDM)
- Finite Element Analysis (FEA)
- FRF Synthesis

